

STRUCTURAL SUBMITTAL PACKAGE

for

Reliable Design

Audio Towers (Indoor)

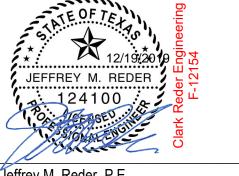
Project Location: US

Event Dates: Varies

Submittal Date: 12/19/2019

Clark Reder Project Number: 19.418.17

Prepared by:



Jeffrey M. Reder, P.E. TX Registration #: 124100



Table of Contents for Structural Submittal Package

Cover Page	1
Table of Contents	2
Project Information	3
Reference Drawings	Appendix A
Calculations	Appendix B

Project Name: Audio Towers (Indoor) CRE Project No: 19.418.17



Project Information

Project Summary

The project referenced by this submittal consists of indoor steel framed audio support towers. The towers will be used at a maximum height of 30 feet for indoor use only.

Scope of Review

Clark Reder Engineering reviewed the capacity of the steel framed towers, headblock and base members. The audio tower was reviewed for a maximum cantilever load of 5,200 pounds per the Client. Rear ballast is required for this installation and can be found in Appendix A.

Conclusions

Our review has concluded that the Audio Tower meets the structural requirements of the 2015 International Building Code, ASCE 7-10, AND ASCE 37-14. The structure shall be used indoors only for a maximum height of 30 feet

Limitations and Exceptions

The scope of review for this submittal is limited to the items listed above. All other temporary or permanent structures on site not specifically referenced above under "Scope of Review" are the responsibility of others.

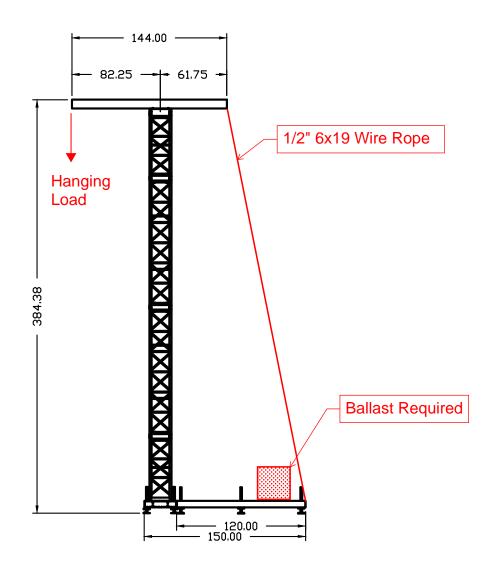
Where the items covered by this submittal are attached to existing structures, it is the responsibility of the engineer of record for those existing structures to review the impact of the elements referenced in this submittal.



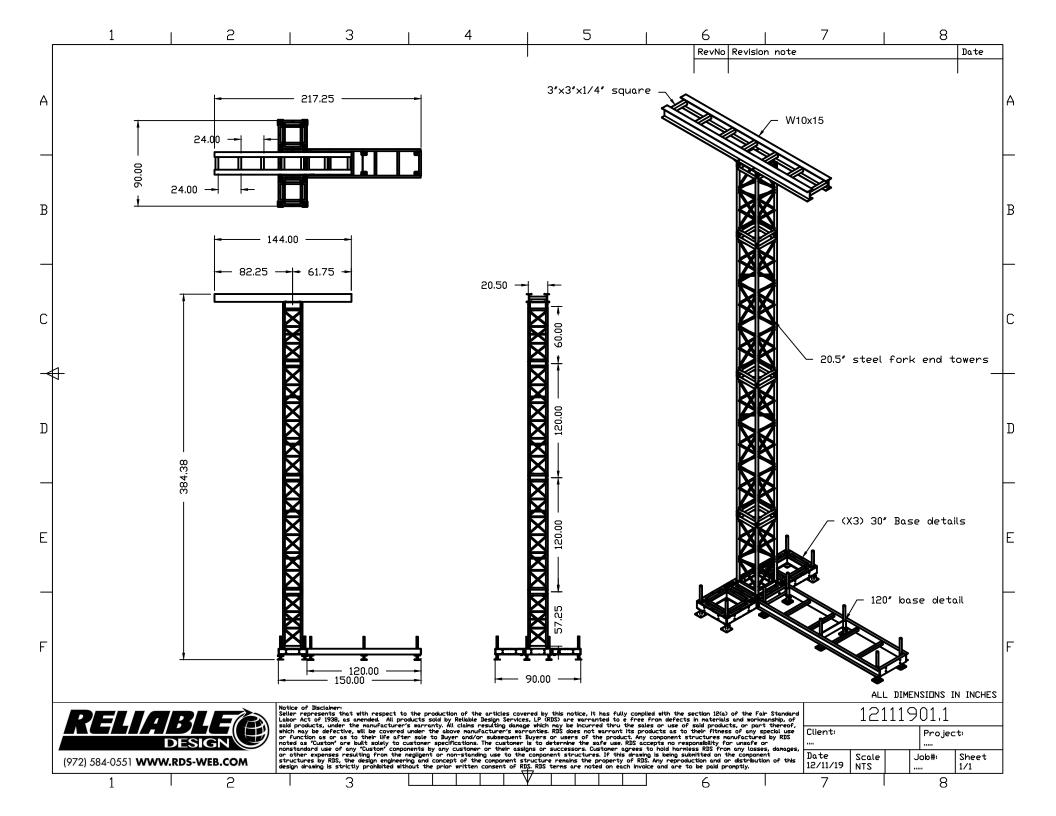
APPENDIX A

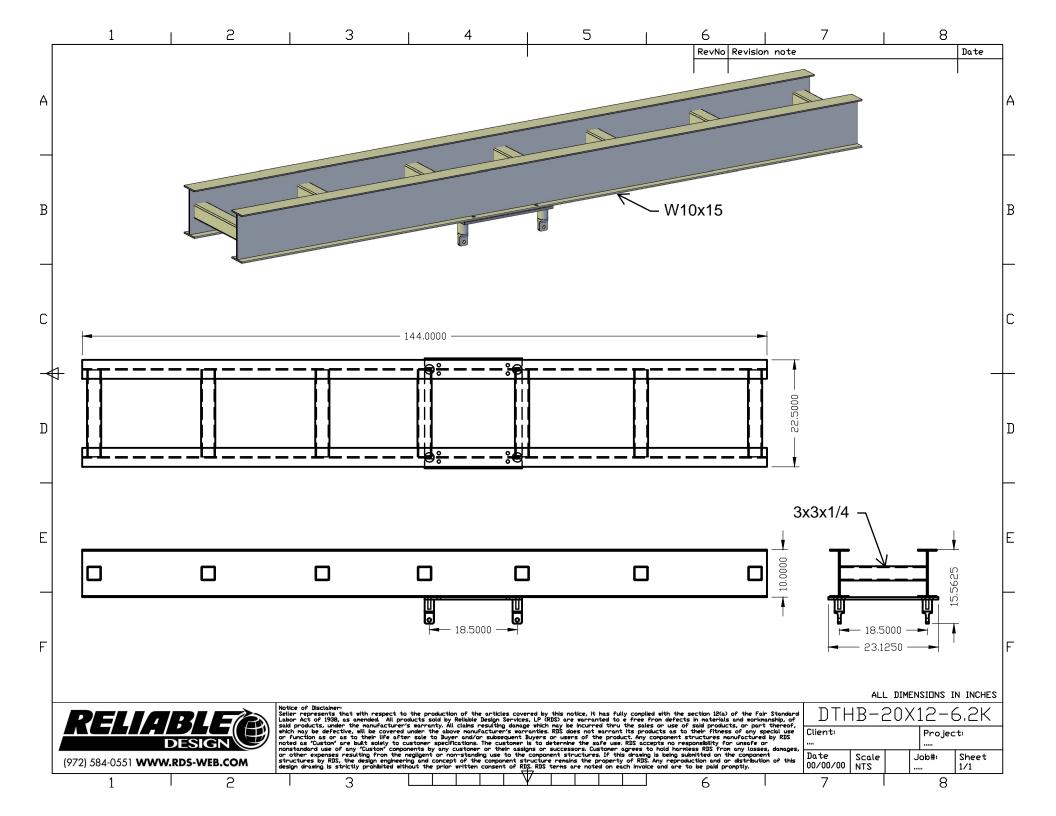


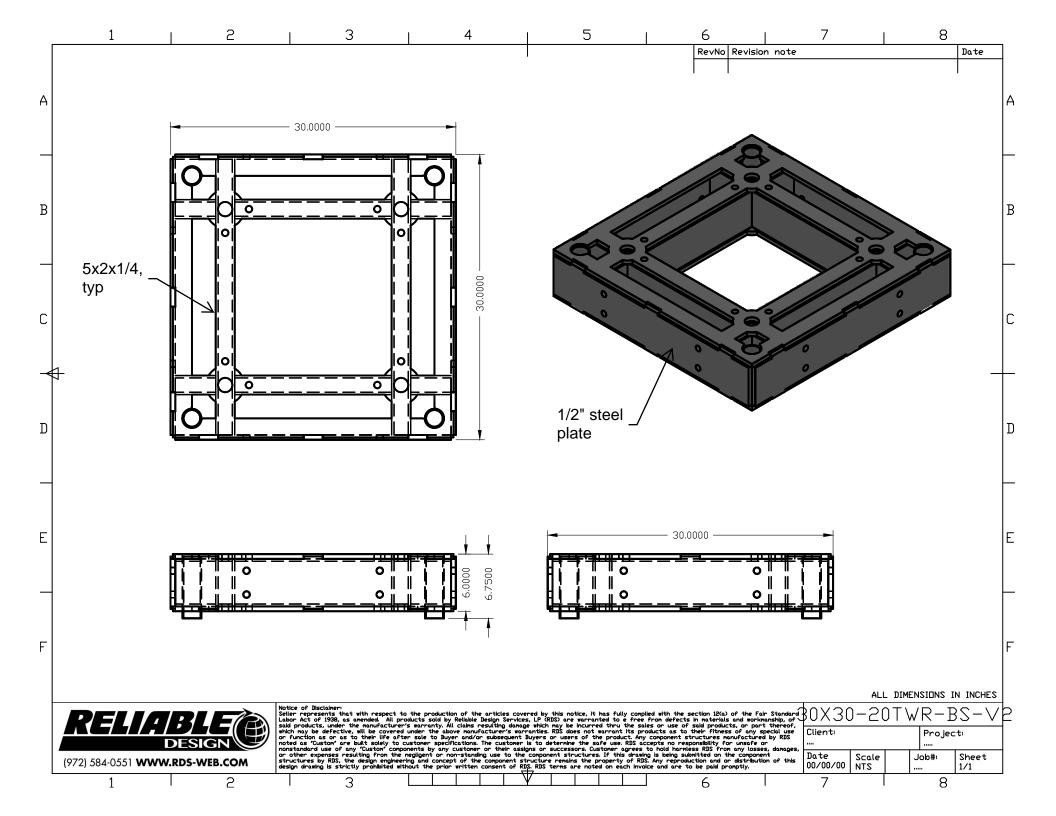
Hanging Load (lb)	Ballast Required (lb)
1000	525
2000	1450
3000	2375
4000	3300
5200	4400

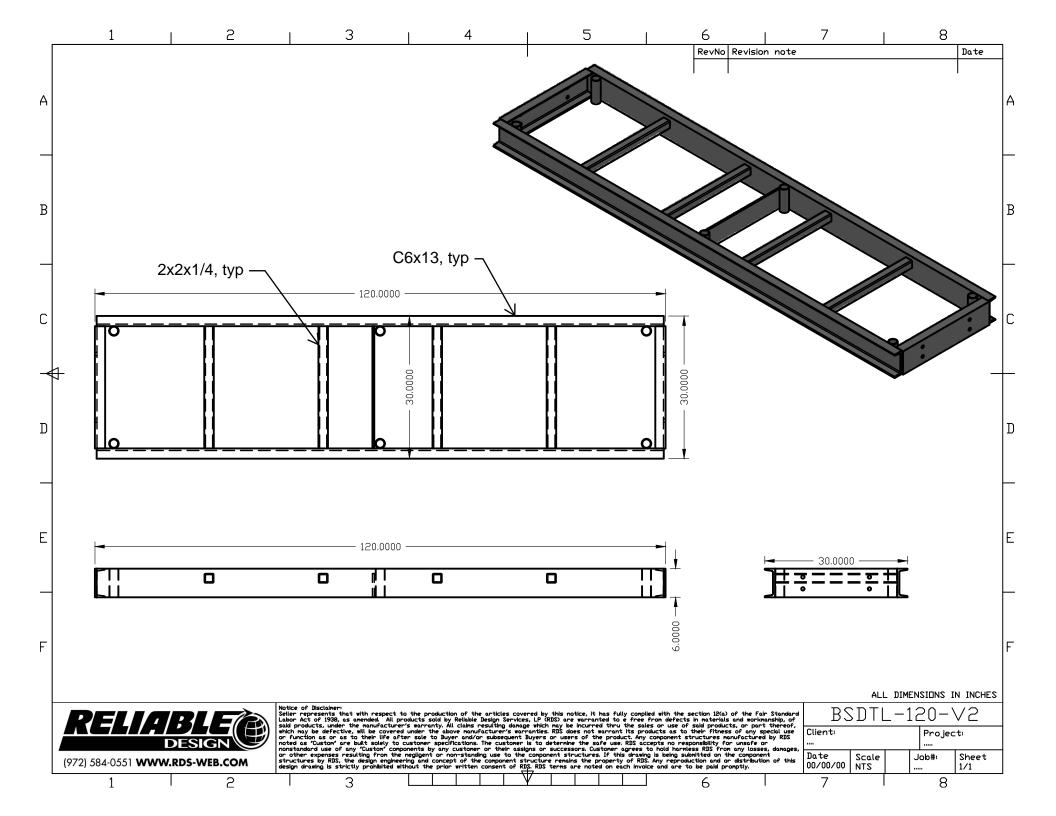


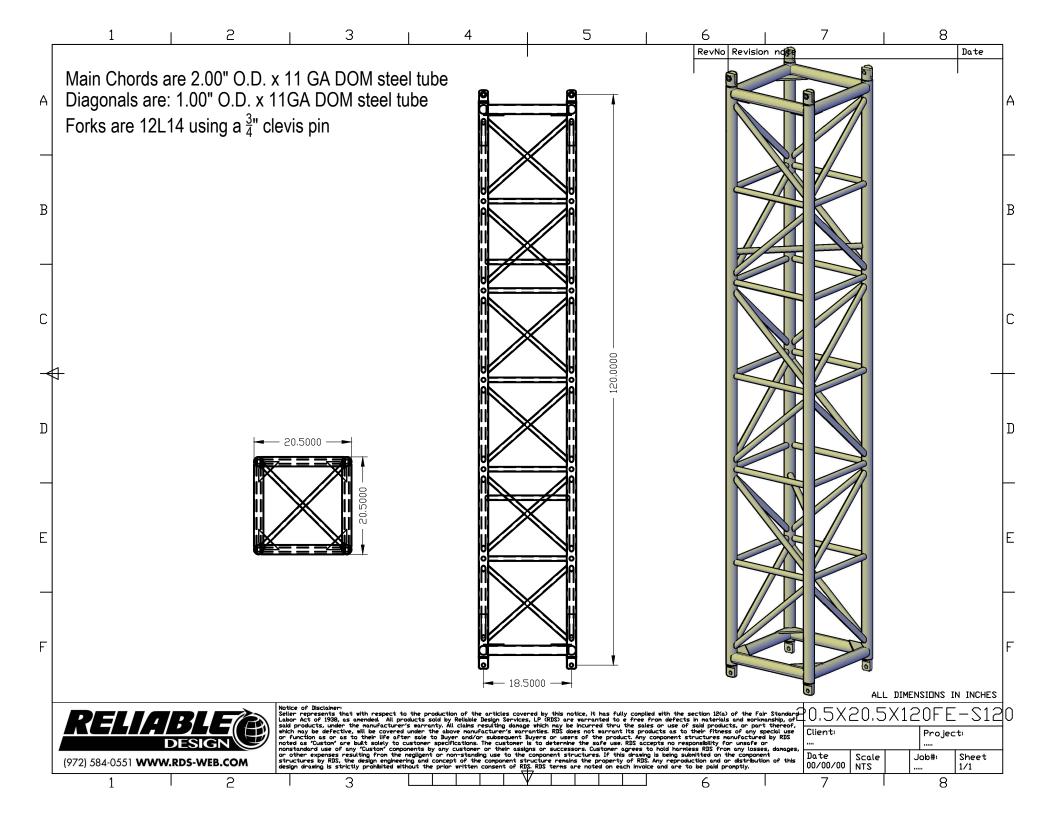
Ballast shown is for indoor use only

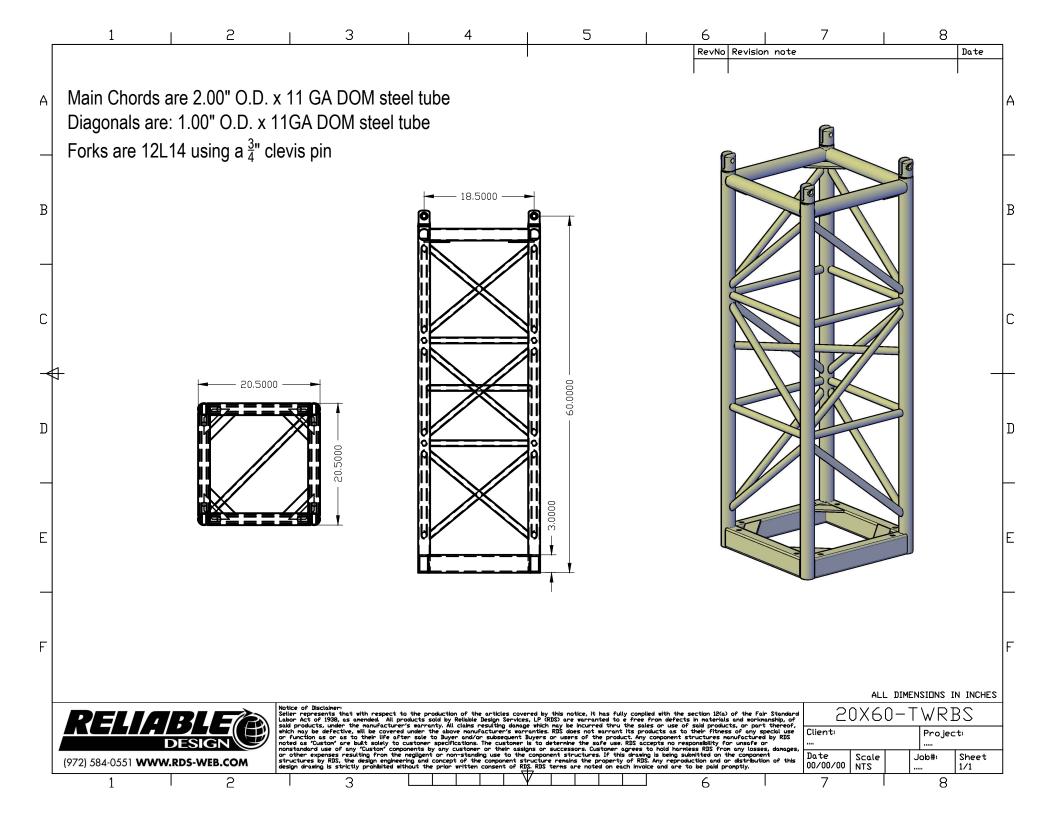














APPENDIX B



Audio Towers

Event Date & Location: Anywhere USA, Indoors, Temporary

Codes and Referenced Standards

- 2015 International Building Code
- Aluminum Design Manual, 2015 ed.
- American Institute of Steel Construction, Steel Construction Manual 15th Edition
- American Society of Civil Engineers 7-10 (ASCE 7-10) "Minimum Design Loads for Buildings and Other Structures"
- American Society of Civil Engineers 37-14 (ASCE 37-14) "Design Loads on Structures During Construction"
- ANSI E 1.21-2013 "Temporary Structures Used for Technical Production of Outdoor Entertainment Events"
- ANSI E 1.2-2012 "Manufacture and Use of Aluminum Trusses and Towers"
- ICC 300-2012, "Bleachers, Folding and Telescopic Seating, and Grandstands"

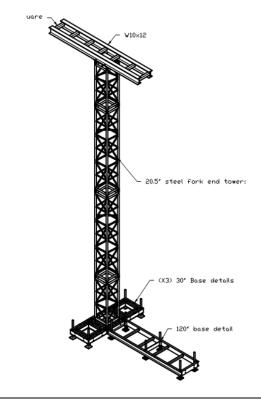
Project Description

The indoor style delay towers will be used to support up to a maximum speaker weight of 5,200 pounds. The tower will be at a maximum height of 30 feet with a 6 ft cantilever to support the audio.

Analysis Assumptions/Design Criteria

Structure is indoors.

Structure will require a rear ballast and guy wire.



 Project:
 Audio
 1 of 4
 Engineer: JMR

 CRE Project #: 19.418.17
 12/17/2019



Gravity Loads

 $\label{eq:parameter} \textbf{Maximum weight of speaker array:} \qquad P_{speaker_max} \coloneqq 5200 \cdot lbf$

Incremental weight of speakers: $P_{speaker 1} := 1000 \cdot lbf$ $P_{speaker 3} := 3000 \cdot lbf$

 $P_{speaker~2} \coloneqq 2000 \cdot lbf \qquad \quad P_{speaker_4} \coloneqq 4000 \cdot lbf$

Weight of tower section: $wt_{10 \text{ tower}} := 242 \cdot lbf \cdot 3 = 726.00 \, lbf$

Weight of 30"x30" base: $wt_{base \ 30} := 195.0962 \cdot in^3 \cdot \gamma_S \cdot 2 \ + \ 73.26 \cdot in^3 \cdot \gamma_S \cdot 4 \ + \ 52.74 \cdot in^3 \cdot \gamma_S \cdot 2 \ ... \ = 296.06 \ lbf$

 $+90.93 \cdot \text{in}^3 \cdot \gamma_{\text{S}} \cdot 2 + 13.58 \cdot \text{in}^3 \cdot \gamma_{\text{S}} \cdot 4 + 4.79 \cdot \text{in}^3 \cdot \left(\gamma_{\text{S}} \cdot 4\right)$

 $\text{Weight of 10 ft base:} \qquad \text{wt}_{base \ 10} \coloneqq 467.81 \cdot \text{in}^3 \cdot \gamma_{\text{S}} \cdot 2 \ + \ 43.57 \cdot \text{in}^3 \cdot \gamma_{\text{S}} \cdot 4 \ + \ 100.1 \cdot \text{in}^3 \cdot \gamma_{\text{S}} \cdot 2 \ \dots \ = \ 392.39 \ \text{lbf}$

 $+48.16 \cdot \text{in}^3 \cdot \gamma_{\text{S}} \cdot 1 + 4.25 \cdot \text{in}^3 \cdot \gamma_{\text{S}} \cdot 6$

Weight of head block: $wt_{head\ block} := 144 \cdot in \cdot 2 \cdot 15 \cdot plf \ + \ 49.26 \cdot in^3 \cdot \gamma_s \cdot 7 = 457.78 \ lbf$





General Overturning

Assume load of array is at tip of tower.

Resisting moment of assemly for tipping caused by audio eccentricity:

$$M_{Resist} \coloneqq \text{wt}_{10_tower} \cdot \frac{18.5 \cdot \text{in}}{2} + \text{wt}_{base_30} \cdot 3 \cdot \frac{18.5 \cdot \text{in}}{2} + \text{wt}_{base_10} \cdot 84.25 \cdot \text{in} = 3999.13 \cdot \text{ft} \cdot \text{lbf}$$

Overturning due to various loads:

$$M_{OT max} := P_{speaker max} \cdot 70.5 \cdot in = 30.55 \cdot ft \cdot kip$$

$$M_{OT\ 1} := P_{speaker\ 1} \cdot 70.5 \cdot in = 5.88 \cdot ft \cdot kip$$

$$M_{OT_2} := P_{speaker_2} \cdot 70.5 \cdot in = 11.75 \cdot ft \cdot kip$$

$$M_{OT_3} := P_{speaker_3} \cdot 70.5 \cdot in = 17.62 \cdot ft \cdot kip$$

$$M_{OT~4} := P_{speaker~4} \cdot 70.5 \cdot in = 23.50 \cdot ft \cdot kip$$

Ballast will be located centered in the rear between the (4) screw jacks:

$$ballast_max := \frac{1.5 \cdot M_{OT_max} - M_{Resist}}{114.0625 \cdot in} = 4400.31 \ lbf$$

ballast_1 :=
$$\frac{1.5 \cdot M_{OT_1} - M_{Resist}}{114.0625 \cdot in} = 506.39 \, lbf$$

$$ballast_2 := \frac{1.5 \cdot M_{OT_2} - M_{Resist}}{114.0625 \cdot in} = 1433.52 \text{ lbf}$$

ballast_3 :=
$$\frac{1.5 \cdot M_{OT_3} - M_{Resist}}{114.0625 \cdot in} = 2360.64 \text{ lbf}$$

$$ballast_4 := \frac{1.5 \cdot M_{OT_4} - M_{Resist}}{114.0625 \cdot in} = 3287.76 \, lbf$$

Project: Audio CRE Project #: 19.418.17

3 of 4

Engineer: JMR 12/17/2019



Consider Lateral Sway - Side to Side

Use 5% of the audio weight.

$$M_{OT maxside} := 5\% \cdot P_{speaker max} \cdot 30 \cdot ft = 7.80 \cdot ft \cdot kip$$

$$M_{OT\ 1side} := 5\% \cdot P_{speaker\ 1} \cdot 30 \cdot ft = 1.50 \cdot ft \cdot kip$$

$$M_{OT\ 2side} := 5\% \cdot P_{speaker\ 2} \cdot 30 \cdot ft = 3.00 \cdot ft \cdot kip$$

$$M_{OT \ 3side} := 5\% \cdot P_{speaker \ 3} \cdot 30 \cdot ft = 4.50 \cdot ft \cdot kip$$

$$M_{OT \ 4side} := 5\% \cdot P_{speaker \ 4} \cdot 30 \cdot ft = 6.00 \cdot ft \cdot kip$$

Resisting moment neglecting rear 10' base

$$M_{Resist_side} := wt_{10_tower} \cdot \frac{85.5 \cdot in}{2} + wt_{base_30} \cdot 3 \cdot \frac{85.5 \cdot in}{2} = 5750.48 \cdot ft \cdot lbf$$

Ballast will be located centered on the 30" base:

$$ballast_max_side := \frac{1.5 \cdot M_{OT_maxside} - M_{Resist_side} - P_{speaker_max} \cdot 42.75 \cdot in}{72.75 \cdot in} = -2074.31 \; lbf =$$

$$ballast_1_side := \frac{1.5 \cdot M_{OT_1side} - M_{Resist_side} - P_{speaker_1} \cdot 42.75 \cdot in}{72.75 \cdot in} = -1165.03 \ lbf$$

$$ballast_2_side := \frac{1.5 \cdot M_{OT_2side} - M_{Resist_side} - P_{speaker_2} \cdot 42.75 \cdot in}{72.75 \cdot in} = -1381.52 \ lbf$$

$$ballast_3_side := \frac{1.5 \cdot M_{OT_3side} - M_{Resist_side} - P_{speaker_3} \cdot 42.75 \cdot in}{72.75 \cdot in} = -1598.02 \text{ lbf}$$

$$ballast_4_side := \frac{1.5 \cdot M_{OT_4side} - M_{Resist_side} - P_{speaker_4} \cdot 42.75 \cdot in}{72.75 \cdot in} = -1814.51 \ lbf$$



Audio Towers - 20.5x20.5

Safety Factors Used in Design

Bending: $\Omega_b := 1.67$ Bearing: $\Omega_{brg} := 2.0$ Pin shear: $\Omega_v := 2.0$ Weld $\Omega_{weld} := 2.0$:

Tension: $\Omega_t := 1.67$ Tensile rupture: $\Omega_{tr} := 2.0$ Shear yielding: $\Omega_{sy} := 1.5$

Compression: $\Omega_c := 1.67$ Shear rupture: $\Omega_{sr} := 2.0$ Block shear: $\Omega_{hs} := 2.0$

Chord member properties - 2"x11 GA (DOM):

Pipe Properties

Pipe diameter: $OD_{chord} := 2in$

Pipe thickness:

 $t_{chord} = 0.12in$

Radius:

 $R_{chord} := \frac{OD_{chord}}{2} = 1 \cdot in$

Pipe yield strength:

 $Fy_{chord} := 75 \cdot ksi$

Inner diameter:

 $ID_{chord} := OD_{chord} - 2 \cdot t_{chord} = 1.76 \cdot in$

Pipe tensile strength: $Fu_{chord} := 85 \cdot ksi$

Moment of inertia:

 $I_{chord} := \frac{\pi \cdot \left(OD_{chord}^{4} - ID_{chord}^{4}\right)}{64} = 0.314 \cdot in^{4}$

Area:

 $A_{chord} := \frac{\pi \cdot \left(OD_{chord}^2 - ID_{chord}^2\right)}{4} = 0.709 \cdot in^2$

Radius of gyration:

 $r_{chord} := \sqrt{\frac{I_{chord}}{A_{chord}}} = 0.666 \cdot in$

Elastic section modulus:

 $S_{\text{chord}} := \frac{\pi \cdot \left(OD_{\text{chord}}^{4} - ID_{\text{chord}}^{4}\right)}{32 \cdot OD_{\text{chord}}} = 0.314 \cdot \text{in}^{3}$

Plastic section moduls:

 $Z_{\text{chord}} := \frac{\left(\text{OD}_{\text{chord}}^3 - \text{ID}_{\text{chord}}^3\right)}{6} = 0.425 \cdot \text{in}^3$

D over t: $D_{t_{chord}} := \frac{OD_{chord}}{t_{chord}} = 16.667$

 $wt_{chord} = 2.412 \cdot plf$

Radius:

 $t_{\text{diag}} := 0.125 \text{in}$

Pipe tensile strength: $Fu_{diag} := 85 \cdot ksi$



Diagonal Member Properties - 1"x11GA (DOM)

$$R_{diag} := \frac{OD_{diag}}{2} = 0.5 \cdot in$$
 Pipe yield strength: Fy_{diag} := 75 · ksi

Pipe thickness:

Inner diameter:
$$ID_{diag} := OD_{diag} - 2 \cdot t_{diag} = 0.75 \cdot in$$

Moment of inertia:
$$I_{diag} := \frac{\pi \cdot \left(OD_{diag}^{4} - ID_{diag}^{4}\right)}{64} = 0.034 \cdot in^{4}$$

Area:
$$A_{diag} := \frac{\pi \cdot \left(OD_{diag}^2 - ID_{diag}^2\right)}{4} = 0.344 \cdot in^2$$

$$\text{Radius of gyration:} \qquad r_{diag} \coloneqq \sqrt{\frac{I_{diag}}{A_{diag}}} = 0.313 \cdot in$$

Elastic section modulus:
$$S_{diag} := \frac{\pi \cdot \left(OD_{diag}^{4} - ID_{diag}^{4}\right)}{32 \cdot OD_{diag}} = 0.067 \cdot in^{3}$$

Plastic section moduls:
$$Z_{diag} := \frac{\left(OD_{diag}^{3} - ID_{diag}^{3}\right)}{6} = 0.096 \cdot in^{3}$$

D over t:
$$D_{tdiag} := \frac{OD_{diag}}{t_{diag}} = 8$$

$$\label{eq:wt_diag} \text{Weight of diagonals:} \qquad wt_{diag} \coloneqq \gamma_s \cdot A_{diag} \qquad \qquad wt_{chord} = 2.412 \cdot plf$$

Horizontal Member Properties - 2"x11 GA (DOM)

Pipe diameter: OD_{horz} :=
$$2in$$
 Pipe thickness: t_{horz} := $0.125in$

Radius:
$$R_{horz} := \frac{OD_{horz}}{2} = 1 \cdot in$$
 Pipe yield strength: $Fy_{horz} := 75 \cdot ksi$

Pipe tensile strength:
$$ID_{horz} := OD_{horz} - 2 \cdot t_{horz} = 1.75 \cdot in$$





Moment of inertia:
$$I_{horz} \coloneqq \frac{\pi \cdot \left(OD_{horz}^{4} - ID_{horz}^{4}\right)}{64} = 0.325 \cdot in^{4}$$

Area:
$$A_{horz} := \frac{\pi \cdot \left(OD_{horz}^2 - ID_{horz}^2\right)}{4} = 0.736 \cdot in^2$$

$$\text{Radius of gyration:} \qquad r_{horz} \coloneqq \sqrt{\frac{I_{horz}}{A_{horz}}} = 0.664 \cdot in$$

Elastic section modulus:
$$S_{horz} := \frac{\pi \cdot \left(OD_{horz}^{4} - ID_{horz}^{4}\right)}{32 \cdot OD_{horz}} = 0.325 \cdot in^{3}$$

Plastic section moduls:
$$Z_{horz} := \frac{\left(OD_{horz}^{3} - ID_{horz}^{3}\right)}{6} = 0.44 \cdot in^{3}$$

D over t:
$$D_{t_{horz}} := \frac{OD_{horz}}{t_{horz}} = 16$$

Weight of diagonals:
$$wt_{horz} := \gamma_s \cdot A_{horz}$$
 $wt_{horz} = 2.506 \cdot plf$

Global Column Section Properties

Truss chord centerline to centerline: $\frac{d_{column} := 18 \cdot in - OD_{chord} = 16 \cdot in}{d_{column}}$

Area of column:
$$A_{column} := 4 \cdot A_{chord}$$
 $A_{column} = 2.835 \cdot in^2$

Moment of inertia of column:
$$I_{column} := 4 \cdot I_{chord} + 4 \cdot A_{chord} \cdot \left(\frac{d_{column}}{2}\right)^2$$

$$I_{column} = 182.696 \cdot in^4$$

Radius of gyration of column:
$$r_{column} := \sqrt{\frac{I_{column}}{A_{column}}}$$
 $r_{column} = 8.028 \cdot in$

$$\begin{aligned} \text{Weight of typical tower:} & \quad \text{wt}_{tower_mid} \coloneqq A_{chord} \cdot \gamma_s \cdot 4 \cdot 116.75 \cdot \text{in} + A_{diag} \cdot \gamma_s \cdot 24 \cdot \text{in} \cdot 24 + A_{diag} \cdot \gamma_s \cdot 20 \cdot 16.5 \cdot \text{in} \\ & \quad + 4 \cdot \text{lbf} \cdot 8 + A_{horz} \cdot \gamma_s \cdot 8 \cdot 16.5 \cdot \text{in} \end{aligned}$$

$$wt_{tower_mid} = 241.693 lbf$$



Column Chord Analysis - 2"x11GA (ERW A53 steel)

Built Up Member Capacity - E.6

Distance between connectors (diagonals): $a_i := 19 \cdot in$

Effective length factor: $K_i := 0.86$ for all other case

 $\label{eq:minimum} \mbox{Minimum radius of gyration on individual component:} \qquad \qquad r_i := r_{chord} = 0.666 \cdot in$

Ratio of a/r: $ratio_a_r := \frac{a_i}{r_i} = 28.527$

Slenderness ratio of built up member = E6-2a.

Column Global Axial Capacity

The column will be checked for the full length capacity of the global section. Use Table 4-22 of the AISC code to determine the available critical stress.

K factor: $k_{column} := 1.25$ vertical diagonal X-bracing

Column length: $L_{column} := 50 \cdot ft$

KL/r $kL_r_{column} := \frac{k_{column} \cdot L_{column}}{r_{column}}$ $kL_r_{column} = 93.427$

 $F_e := \frac{\pi^2 \cdot E_s}{\left(kL_{r_{column}}\right)^2} = 32.791 \cdot ksi$

 $F_{\text{cr_column}} := \begin{bmatrix} 0.658 \\ \hline F_{e} \\ \end{bmatrix} \cdot F_{\text{ychord}} \text{ if } kL_r_{\text{column}} < 4.71 \cdot \sqrt{\frac{E_{s}}{F_{\text{ychord}}}} \\ = 28.758 \cdot ksi \\ 0.877 \cdot F_{e} \text{ if } kL_r_{\text{column}} > 4.71 \cdot \sqrt{\frac{E_{s}}{F_{\text{ychord}}}} \end{bmatrix}$

Allowable global axial load: $P_{a_column} := F_{cr_column} \cdot A_{column}$ $P_{a_column} = 81.527 \cdot kip$



Column Local Axial Capacity

The chords are braced by diagonals on all sides. The local axial load capacity of each chord will be determined and compared to the global capacity.

K factor:

$$k_{local} := 1.0$$

Column length:

$$L_{local} := 18 \cdot in$$

$$\label{eq:kl_r} \text{KL/r:} \qquad \text{kL_r}_{local} \coloneqq \frac{\text{k}_{local}\text{`}L_{local}}{\text{r_{chord}}}$$

$$kL_{-}r_{local} = 27.026$$

$$F_{\text{New}} := \frac{\pi^2 \cdot E_{\text{S}}}{\left(\frac{k_{\text{local}} \cdot L_{\text{local}}}{r_{\text{chord}}}\right)^2} = 391.872 \cdot ksi$$

$$F_{cr_local} := \begin{bmatrix} 0.658 & \frac{\left(\frac{Fy_{chord}}{F_e}\right)}{F_e} \\ 0.658 & \frac{\left(\frac{Fy_{chord}}{F_e}\right)}{F_{chord}} & \text{if } \frac{k_{local} \cdot L_{local}}{r_{chord}} < 4.71 \cdot \sqrt{\frac{E_s}{Fy_{chord}}} \\ 0.877 \cdot F_e & \text{if } \frac{k_{local} \cdot L_{local}}{r_{chord}} > 4.71 \cdot \sqrt{\frac{E_s}{Fy_{chord}}} \end{bmatrix} = 69.226 \cdot ksi$$

Allowable global axial load:

$$P_{a local} := F_{cr local} \cdot A_{chord} \cdot 4$$

$$P_{a local} = 196.255 \cdot kip$$

Per E.6.2 the effective slenderness ratio of the component shapes between fasteners shall not exceed 3/4 x the governing slenderness ratio of the built up member.

$$kL_r_{local} = 27.026 \quad \text{less than} \qquad kL_r_{column} \cdot 0.75 = 70.07 \quad \text{Acceptable}$$



Allowable Concentrated Force On Chord Member

Check Flexural Capacity Chord Between Panel Points

Determine the maximum concentrated load that the chord can resist.

Check limiting width to thickness ratios for compression elements using Table B4.1.

$$Dt_flex := 0.11 \cdot \frac{E_s}{Fy_{chord}}$$

$$Dt_flex = 42.533$$

$$Dt_flex = 42.533$$
 > $D_t_{chord} = 16.667$ compact

$$Dt_comp := 0.07 \cdot \frac{E_s}{Fy_{chord}} \qquad \qquad Dt_comp = 27.067 \qquad \text{>} \qquad D_t_{chord} = 16.667 \quad \underset{t}{\textbf{compac}}$$

$$Dt_comp = 27.067$$

$$D_{-t_{chord}} = 16.667$$
 compact

$$Dt_F8 := 0.45 \cdot \frac{E_s}{Fy_{chord}}$$
 $Dt_{chord} = 16.667$

$$Dt_comp = 27.067$$

$$D_{topord} = 16.667$$

Local buckling does not apply as section is compact. Yielding controls the flexural strength of the member.

Nominal moment capacity:

$$M_n := Fy_{chord} \cdot Z_{chord}$$

$$M_n = 2.654 \cdot \text{ft} \cdot \text{kip}$$

Allowable moment capacity:

$$M_{a_chord} := \frac{M_n}{\Omega_b}$$

$$M_{a chord} = 1.589 \cdot ft \cdot kip$$

Allowable concentrated load:

$$P_{conc_chord} \coloneqq \frac{8 \cdot M_{a_chord}}{L_{local}}$$

$$P_{conc_chord} = 8.477 \cdot kip$$

asssumes load applied at midspan between diagonals and fixed end beam condition

Allowable Axial Tension Capacity

Tensile yielding on gross section:

$$P_{n_y} := Fy_{chord} \cdot A_{chord}$$

$$P_{n_y} = 53.156 \cdot \text{kip}$$

Allowable tension capacity:

$$P_{\text{ta_chord}} := 4 \cdot \frac{P_{\text{n_y}}}{\Omega_{\text{t}}}$$

$$P_{ta_chord} = 127.319 \cdot kip$$



Diagonal Analysis - 1"x1/8" (ERW A53)

Diagonal Axial Capacity (Compression)

Per AISC Chapter E, Section 6 the lacing shall be proportioned to provide a shearing strength normal to the axis of the member equal to 2 percent of the available compressive strength of the member.

K factor: $k_{\text{diag}} := 1.0$

Diagonal length: $L_{diag} := 19.8 \cdot in$

 $\label{eq:KL/r:kL_rdiag} \text{KL/r:} \qquad kL_r_{diag} := \frac{k_{diag} \cdot L_{diag}}{r_{diag}}$

 $kL_r_{diag} = 63.36$ less than 140 - OK

Critical stress from Table 4-22: $F_{cr\ diag} := 16.5 \cdot ksi$

Allowable diagonal axial load: $P_{a_diag} := F_{cr_diag} \cdot A_{diag}$ $P_{a_diag} = 5.67 \cdot kip$

By inspection of the loads, the lacing meets the minimum requirements of the AISC code.



Forked End Capacity

The towers are connected using forked end connectors. The material strength of the pins and forks are as follows 12L14 steel connected with (2) spirol pins to the actual truss.

Pin yield stress: $Fy_{pin} := 60 \cdot ksi = 60 \cdot ksi$

Pin tensile stress: $Fu_{pin} := 78 \cdot ksi = 78 \cdot ksi$

The typical tower truss is connected using 0.75 diameter 12L14 steel pins. The pin material is not referenced of AISC 13th edition. However, use Table J3.2 to determine allowable solution in lieu of other information.

Table J3.2 (sim to A325): $F_{nv pin} := Fu_{pin} \cdot 0.5$ $F_{nv pin} = 39 \cdot ksi$

Pin diameter: $d_{pin} := 0.75 \cdot in$

Area of pin: $A_{pin} := \frac{d_{pin}^2 \cdot \pi}{4}$ $A_{pin} = 0.442 \cdot in^2$

Allowable pin shear at fork: $R_{n_over_\Omega.bolt} := \frac{F_{nv_pin} \cdot A_{pin} \cdot 2}{2.0} \quad R_{n_over_\Omega.bolt} = 17.23 \cdot kip \quad \text{ bolt in double shear}$

Total fork bearing capacity: $R_{brg\ max} := 4 \cdot R_{n\ over\ \Omega.bolt}$ $R_{brg\ max} = 68.919 \cdot kip$

Forked Properties

1.59" diameter 12L14 material solid round. End machined to a double clevis configuration or a single clevis. The forked end does not meet the dimensional requirements of a pin connected member per AISC. Therefore consider it a bolted connection.

Outside diameter of spigot: $OD_{fork} := 1.6in$

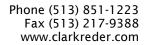
Wal thickness: $t_{fork} := 0.35 \cdot in$

Inside diameter: $ID_{fork} := OD_{fork} - 2 \cdot t_{fork}$ $ID_{fork} = 0.9 \cdot in$

Area of spigot at inserted section: $A_{fork} := \frac{\pi \cdot \left(OD_{fork}^{2} - ID_{fork}^{2}\right)}{4}$ $A_{fork} = 1.374 \cdot in^{2}$

Moment of inertia at inserted section: $I_{fork} := \frac{\pi \cdot \left(OD_{fork}^{4} - ID_{fork}^{4}\right)}{64}$ $I_{fork} = 0.289 \cdot in^{4}$

Section modulus: $S_{fork} := \frac{\pi \cdot \left(OD_{fork}^{4} - ID_{fork}^{4}\right)}{325 \cdot OD_{fork}}$ $S_{fork} = 0.036 \cdot in^{3}$





Male Plate Bending:

Plate Tension Capacity (AISC D5):

Safety factor for tensile yielding: $\Omega_{ty} := 1.67$ Safety factor for flexure: $\Omega_{ty} := 1.67$

Safety factor for tensile rupture $\Omega_{\text{Mtx}} = 2.0$ Safety factor for tension: $\Omega_{\text{mtx}} = 1.67$

Safety factor for shear yielding: $\Omega_{\text{NSW}} = 1.5$ Safety factor for bearing: $\Omega_{\text{NSW}} = 2.0$

Safety factor for shear rupture: $\Omega_{\text{No.}} = 2$ Safety factor for compression: $\Omega_{\text{No.}} = 1.67$

Safety factor for weld: $\Omega_{\text{weld}} = 2.0$ Safety factor for bearing: $\Omega_{\text{byex}} = 2.0$

spigot plate properties

Width of plate: b := 1.97in

Thickness of plate: t := 0.625in

Span of plate: $L_p := 0.96in$

area of plate: $A_p := b \cdot t$ $A_p = 1.231 \cdot in^2$

Plastic Section modulus: $Z_x := \frac{t \cdot b^2}{4}$ $Z_x = 0.606 \cdot in^3$ $Z_y := \frac{b \cdot t^2}{4}$ $Z_y = 0.192 \cdot in^3$

Allowable moments: $M_{y_allow} := \frac{Fy_{pin} \cdot Z_y}{\Omega_L} \qquad \qquad M_{y_allow} = 0.576 \cdot kip \cdot ft$

 $M_{x_allow} := \frac{Fy_{pin} \cdot Z_x}{\Omega_L}$ $M_{x_allow} = 1.816 \cdot kip \cdot ft$

Allowable Axial Compression

Fork plate

moment of inertia weak axis: $I_y := \frac{b \cdot t^3}{12}$ $I_y = 0.04 \cdot in^4$

radius of gyration weak axis: $r_y := \sqrt{\frac{I_y}{A_p}}$ $r_y = 0.18 \cdot in$

Effective length factor: $K_{eff, sec} := 2$

Slenderness factor: $kl_r_{sec} := \frac{K_{eff_sec} \cdot L_p}{r_v}$ $kl_r_{sec} = 10.642$



Elastic critical buckling value:
$$F_{e.sec} := \frac{\pi^2 \cdot 29000 \cdot ksi}{kl r_{ceo}^2}$$

$$F_{e.sec} = 2527.401 \cdot ksi$$

$$F_{\text{cr.sec}} := \text{if} \left[kl_r_{\text{sec}} > 4.71 \cdot \sqrt{\frac{29000 \cdot \text{ksi}}{\text{Fy}_{\text{pin}}}}, 0.877 \cdot F_{\text{e.sec}}, \left(\frac{\frac{\text{Fy}_{\text{pin}}}{\text{F}_{\text{e.sec}}}}{0.658} \right) \cdot \text{F } F_{\text{cr.sec}} = 59.407 \cdot \text{ksi} \right]$$

$$P_{\text{ca.sec}} := \frac{F_{\text{cr.sec}} \cdot A_p}{\Omega_c}$$

$$P_{\text{ca.sec}} = 43.799 \cdot \text{kip}$$

$$P_{ca} := 4 \cdot P_{ca,sec}$$

$$P_{ca} = 175.197 \cdot kip$$

Fork Tension Capacity

The pin diameter is 0.75" and the hole size is slightly bigger. The difference is less than 1/32". Per AISC the the diameter of the pin hole shall shall not be more than 1/32" greater than the pin. Use section D5 for pin connected members.

Bearing strength at bolt holes (AISC J7)

Projected bearing area: $A_{pb} := t \cdot d_{pin}$

Available bearing strength:
$$R_{a_bearing} := \frac{1.8 \cdot Fy_{pin} \cdot A_{pb}}{\Omega_{brg}}$$

$$R_{a_bearing} = 25.313 \cdot kip$$

Plate Tension Capacity (AISC J4):

$$P_{ny_plate} := \frac{Fy_{pin} \cdot b \cdot t}{\Omega_{ty}}$$

$$P_{ny_plate} = 44.237 \cdot kip$$

$$P_{n_tr} := \frac{Fu_{pin} \cdot \left[0.85 \cdot \left(b - \frac{13}{16} \cdot in \right) \cdot t \right]}{\Omega_{tr}}$$

$$P_{n tr} = 23.982 \cdot kip$$

$$P_{\text{sy_plate}} := \frac{0.6 \cdot Fy_{\text{pin}} \cdot (b \cdot t)}{\Omega_{\text{sv}}}$$

$$P_{\text{sy_plate}} = 29.55 \cdot \text{kip}$$



Shear rupture (AISC J4-4):

$$P_{sr_plate} := \frac{0.6 \cdot Fu_{pin} \cdot \left[2 \cdot t \cdot \left(1 \cdot in + \frac{d_{pin}}{2} \right) \right]}{\Omega_{sr}}$$

 $P_{sr_plate} = 40.219 \cdot kip$

The forked connection is controlled by the pin shear.

Allowable axial tension due to forked connection:

$$P_{a \text{ fork}} := (4 \cdot R_{n \text{ over } \Omega.bolt})$$

$$P_{a \text{ fork}} = 68.919 \cdot \text{kip}$$

Spirol Coil Spring Pins

The forks are connected to the aluminum with Spirol Pins. The Spirol pins are coiled 3/8" diameter pins located 1" apart with a 1/2" edge distance to the edge of steel. Use a factor of safety of 2 on the coil pins.

Capacity of coiled spirol pin in double shear:

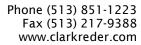
$$P_{allow coil pin} := 12.6 \cdot kip$$

There are two pins therefore allowable load is

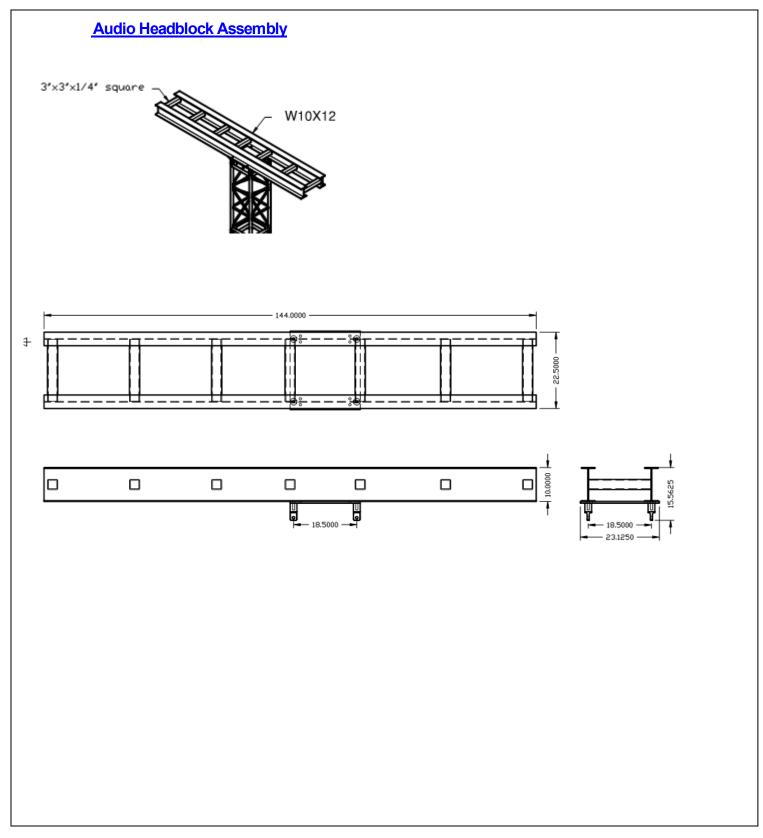
$$P_{allow_coil} := \frac{2 \cdot P_{allow_coil_pin}}{2}$$

$$P_{allow coil} = 12.6 \cdot kip$$

There are (4) chords $P_{allow_coil} \cdot 4 = 50400 \, lbf$









Headblock Beam

Standard Beam AISC Parameters:

E:= 29000·ksi
$$K_{XX}:= 1.0$$
 $K_{XX}:= 1.0$ $K_{XX}:= 1.0$

Beam B := 1 span :=
$$144 \cdot \text{in}$$
 bm start := $0 \cdot \text{plf}$

$$\mathbf{p}_{\mathbf{X}} \coloneqq \begin{pmatrix} \mathbf{0} \cdot \mathbf{kip} \\ \mathbf{0} \cdot \mathbf{kip} \\ \mathbf{0} \cdot \mathbf{kip} \\ \mathbf{0} \cdot \mathbf{kip} \end{pmatrix} \mathbf{d}_{\mathbf{p}\mathbf{X}} \coloneqq \begin{pmatrix} \mathbf{0} \cdot \mathbf{5} \cdot \mathbf{span} \\ 2\mathbf{0} \cdot \mathbf{ft} \\ \mathbf{0} \cdot \mathbf{ft} \\ \mathbf{0} \cdot \mathbf{klf} \\ \mathbf{0} \cdot \mathbf{klf} \end{pmatrix} \mathbf{w}_{\mathbf{1}\mathbf{X}} \coloneqq \begin{pmatrix} \mathbf{bm}_{\mathbf{w}\mathbf{t}} \\ \mathbf{0} \cdot \mathbf{klf} \\ \mathbf{0} \cdot \mathbf{klf} \\ \mathbf{0} \cdot \mathbf{klf} \end{pmatrix} \mathbf{d}_{\mathbf{w}\mathbf{1}\mathbf{X}} \coloneqq \begin{pmatrix} \mathbf{bm}_{\mathbf{w}\mathbf{t}} \\ \mathbf{0} \cdot \mathbf{ft} \\ \mathbf{0} \cdot \mathbf{ft} \\ \mathbf{0} \cdot \mathbf{ft} \end{pmatrix} \mathbf{w}_{\mathbf{2}\mathbf{X}} \coloneqq \begin{pmatrix} \mathbf{bm}_{\mathbf{w}\mathbf{t}} \\ \mathbf{0} \cdot \mathbf{klf} \\ \mathbf{0} \cdot \mathbf{klf} \\ \mathbf{0} \cdot \mathbf{ft} \end{pmatrix}$$

$$\begin{pmatrix} RL_{x_B} \\ RR_{x_B} \\ M_{\text{MMB}} \end{pmatrix} := \begin{pmatrix} R_L \left(p_x, d_{px}, w_{1x}, w_{2x}, d_{w1x}, d_{w2x}, span \right) \\ R_R \left(p_x, d_{px}, w_{1x}, w_{2x}, d_{w1x}, d_{w2x}, span \right) \\ Mom \left(p_x, d_{px}, w_{1x}, w_{2x}, d_{w1x}, d_{w2x}, span \right) \end{pmatrix} \quad RL_{x_B} = 0 \cdot \text{kip} \quad RR_{x_B} = 0 \cdot \text{kip} \quad M_{x_B} = 0 \cdot \text{ft-kip}$$

Analysis Loads:
$$M_X := \frac{5.2 \cdot \text{kip}}{2} \cdot 72 \cdot \text{in}$$
 $M_X := 0 \cdot \text{ft-kip}$ $M_X := 0 \cdot \text{ft-kip}$ $M_X := 0 \cdot \text{kip}$ M

AISC Design Parameters:
$$F_{W} := 50 \cdot ksi$$
 $F_{W} := span$ $F_{W} := span$ $F_{W} := 72 \cdot in$

Member Specification: PROF := "W10X15"

$$A := \text{Results} \left(\text{PROF}, L_x, K_x, L_v, K_v, L_b, C_b, C_{mx}, C_{mv}, M_x, M_v, P_{ax}, V_x, \text{Ratio}, F_v, E \right)$$

$$A = \begin{pmatrix} "W10X15" & "PASS" & "fa_comp (ksi) = " & 0 & "Fa_comp (ksi) = " & 4.725 & "Ix (in^4) = " & 68.9 \\ "IntActRatio = " & 0.55 & "fbx (ksi) & = " & 13.565 & "Fbx (ksi) & = " & 24.656 & "Iy (in^4) = " & 2.89 \\ "Shear Ratio = " & 0 & "fby (ksi) & = " & 0 & "Fby (ksi) & = " & 37.5 & "Fy (ksi) = " & 50 \end{pmatrix}$$

The W10X15 can safely support the weight of the array (5200#/2)



3x3x1/4

Beam

$$B := 1$$

$$\mathbf{p}_{\mathbf{X}} := \begin{pmatrix} 5.2 \cdot \text{kip} \\ 0 \cdot \text{kip} \\ 0 \cdot \text{kip} \end{pmatrix} \mathbf{d}_{\mathbf{p}\mathbf{X}} := \begin{pmatrix} 0 \\ 0 \\ 0 \end{pmatrix}$$

$$\mathbf{w}_{1x} := \begin{pmatrix} \mathbf{bm}_{wt} \\ 0 \cdot \mathbf{klf} \\ 0 \cdot \mathbf{klf} \\ 0 \cdot \mathbf{klf} \end{pmatrix}$$

$$d_{w1x} := \begin{pmatrix} 0 \cdot ft \\ 0 \cdot ft \\ 0 \cdot ft \\ 0 \cdot ft \end{pmatrix}$$

$$\mathbf{w}_{2x} := \begin{pmatrix} \mathbf{bm}_{wt} \\ \mathbf{0} \cdot \mathbf{klf} \\ \mathbf{0} \cdot \mathbf{klf} \\ \mathbf{0} \cdot \mathbf{klf} \end{pmatrix}$$

$$\mathbf{v}_{2\mathbf{x}} := \begin{pmatrix} \mathbf{bm}_{\mathbf{wt}} \\ \mathbf{0} \cdot \mathbf{klf} \\ \mathbf{0} \cdot \mathbf{klf} \\ \mathbf{0} \cdot \mathbf{klf} \end{pmatrix} \mathbf{d}_{\mathbf{w}2\mathbf{x}} := \begin{pmatrix} \mathbf{span} \\ \mathbf{span} \\ \mathbf{0} \cdot \mathbf{ft} \\ \mathbf{0} \cdot \mathbf{ft} \\ \mathbf{0} \cdot \mathbf{ft} \end{pmatrix}$$

$$\begin{pmatrix} RL_{x_B} \\ RR_{x_B} \\ M_{\text{MMB}} \end{pmatrix} := \begin{pmatrix} R_L \Big(p_x, d_{px}, w_{1x}, w_{2x}, d_{w1x}, d_{w2x}, span \Big) \\ R_R \Big(p_x, d_{px}, w_{1x}, w_{2x}, d_{w1x}, d_{w2x}, span \Big) \\ Mom \Big(p_x, d_{px}, w_{1x}, w_{2x}, d_{w1x}, d_{w2x}, span \Big) \end{pmatrix} \quad RL_{x_B} = 2.6 \cdot \text{kip} \quad RR_{x_B} = 2.6 \cdot \text{kip} \quad M_{x_B} = 2.004 \cdot \text{ft} \cdot \text{kip}$$

$$RL_{x_B} = 2.6 \cdot \text{kip}$$
 $RR_{x_B} = 2.6 \cdot \text{kip}$

$$M_{x_B} = 2.004 \cdot \text{ft} \cdot \text{kip}$$

Analysis Loads:

$$M_{X} := \left(M_{X_{B}}\right)$$

$$M_{\text{w}} = 0 \cdot \text{ft} \cdot \text{kip}$$

$$P_{\text{NAXA}} = 0 \cdot \text{kip}$$

$$\mathbf{M}_{\mathbf{X}} \coloneqq \left(\mathbf{M}_{\mathbf{X}_{\mathsf{B}}}\right) \qquad \qquad \underbrace{\mathbf{M}_{\mathbf{X}_{\mathsf{B}}}}_{\mathsf{M}} \coloneqq 0 \cdot \mathsf{ft} \cdot \mathsf{kip} \qquad \underbrace{\mathbf{N}_{\mathsf{M}}}_{\mathsf{M}} \coloneqq 0 \cdot \mathsf{kip} \qquad \underbrace{\mathbf{N}_{\mathsf{M}}}_{\mathsf{M}} \coloneqq \mathsf{Maximum}\left(\mathbf{R}\mathbf{R}_{\mathbf{X}_{\mathsf{B}}}, \mathbf{R}\mathbf{L}_{\mathbf{X}_{\mathsf{B}}}\right)$$

AISC Design Parameters:

$$F_{\text{AAAA}} := 46 \cdot \text{ks}$$

$$L_{\infty} := spa$$

$$L_{\infty} := span$$

$$F_{W}:=46 \cdot \text{ksi}$$
 $F_{W}:=\text{span}$ $F_{W}:=\text{span}$ $F_{W}:=72 \cdot \text{in}$

Member Specification: PROF := "TS3X3X1/4"

$$A := \text{Results}(\text{PROF}, L_x, K_x, L_v, K_v, L_b, C_b, C_{mx}, C_{mv}, M_x, M_v, P_{ax}, V_x, Ratio, F_v, E)$$

$$A = \begin{pmatrix} "TS3X3X1/4" & "PASS" & "fa_comp (ksi) = " & 0 & "Fa_comp (ksi) = " & 26.398 & "Ix (in^4) = " & 3.16 \\ "IntActRatio = " & 0.377 & "fbx (ksi) & = " & 11.452 & "Fbx (ksi) & = " & 30.36 & "Iy (in^4) = " & 3.16 \\ "Shear Ratio = " & 0.094 & "fby (ksi) & = " & 0 & "Fby (ksi) & = " & 30.36 & "Fy (ksi) = " & 46 \end{pmatrix}$$

The HSS3x3x1/4 can support a load of 5200#.

$$P_{\text{weld}} := 3.0.92 \cdot \frac{\text{kip}}{\text{in}} \cdot 12 \cdot \text{in} = 33120 \text{ lbf}$$

The weld is adequate



Plate and Bolted Connections

Check 1/2" aluminum plate as well as connection bolts.

Bar Properties

Bar Depth: $d := 2.75 \cdot \text{in} + 1.25 \cdot \text{in} \cdot 2 = 5.25 \text{ in}$

Bar thickness: t := 0.5in

Aluminum alloy:

Alloy := 6061-T6

Unbraced length (include K-factor):

 $L_{bx} := 6in$

 $L_{by} := 6in$

Pipe Cross Section Properties

Area: Area := $d \cdot t = 2.625 \cdot in^2$

Elastic section $S_X := \frac{t \cdot d^2}{6} = 2.297 \cdot in^3$

 $S_y := \frac{t^2 \cdot d}{6} = 0.219 \cdot in^3$

Moment of inertia: $I_X := \frac{t \cdot d^3}{12} = 6.029 \cdot in^4$

 $I_y := \frac{t^3 \cdot d}{12} = 0.055 \cdot in^4$

Radius of gyration: $r_x := \sqrt{\frac{I_x}{Area}} = 1.516 \cdot in$

 $r_y := \sqrt{\frac{I_y}{Area}} = 0.144 \cdot in$

Alloy Properties

▶ Safety Factors

▶ Buckling Constants



 $F_{b_F812_1} = 27.576 \cdot ksi$

 $F_{b F812_{2}} = 30.585 \cdot ksi$

Unwelded Welded

Ultimate tensile strength: $F_{tu} = 42 \cdot ksi$ $F_{tu} = 42 \cdot ksi$

 $F_{tv} = 35 \cdot ksi$ $F_{tv} = 35 \cdot ksi$ Tensile yield strength:

 $F_{cv} = 35 \cdot ksi$ $F_{cv} = 35 \cdot ksi$ Compressive yield strength:

 $F_{sv} = 21 \cdot ksi$ $F_{sv} = 21 \cdot ksi$ Shear yield strength:

Modulus of elasticity: $E_a = 10100 \cdot ksi$

Flexure Capacity

Section F.8.1.2:

Welded or unwelded:

Tensile rupture:

Moment Tensile yielding: capacity:

 $F_{b_F812_1} := \frac{1}{\Omega_{by}} \cdot \begin{vmatrix} 1.3 \cdot F_{ty} & \text{if Weld}_{F812} = 1 \\ 1.3 \cdot F_{tyw} & \text{otherwise} \end{vmatrix}$

 $F_{b_F812_2} := \frac{1}{\Omega_{bu}} \cdot \left| 1.42 \cdot \frac{F_{tu}}{k_t} \right| \text{ if Weld}_{F812} = 1$ $1.42 \cdot \frac{F_{tuw}}{k_t} \text{ otherwise}$

 $M_{n F812x} := min(F_{b F812}) \cdot S_x$ Allowable flexural $M_{n F812x} = 63.34 \cdot \text{kip} \cdot \text{in}$ capacity:

> $M_{n F812v} := min(F_{b F812}) \cdot S_{v}$ $M_{n F812y} = 6.03 \cdot \text{kip} \cdot \text{in}$

Section F.4.1:

Welded or unwelded:

 $F_{b_F41_1} := \frac{1}{\Omega_{by}} \cdot \begin{bmatrix} 1.3 \cdot F_{cy} & \text{if Weld}_{F41} = 1 \\ 1.3 \cdot F_{cyw} & \text{otherwise} \end{bmatrix}$ Moment Compressive $F_{b_F41_1} = 27.576 \cdot ksi$ capacity: yielding:



Tensile yielding:

$$F_{b_F41_2} := \frac{1}{\Omega_{by}} \cdot \begin{vmatrix} 1.3 \cdot F_{ty} & \text{if Weld}_{F41} = 1\\ 1.3 \cdot F_{tyw} & \text{otherwise} \end{vmatrix}$$

$$F_{b_F41_2} = 27.576 \cdot ksi$$

Tensile rupture:

$$F_{b_F41_3} := \frac{1}{\Omega_{bu}} \cdot \begin{vmatrix} 1.42 \cdot \frac{F_{tu}}{k_t} & \text{if Weld}_{F41} = 1 \\ 1.42 \cdot \frac{F_{tuw}}{k_t} & \text{otherwise} \end{vmatrix}$$

$$F_{b_F41_3} = 30.585 \cdot ksi$$

Allowable flexural capacity:

$$M_{n_F41x} := min(F_{b_F41}) \cdot S_x$$

$$\mathbf{M}_{n_F41y} \coloneqq \min(\mathbf{F}_{b_F41}) \cdot \mathbf{S}_{y}$$

$$M_{n F41x} = 63.34 \cdot \text{kip} \cdot \text{in}$$

$$M_{n F41y} = 6.03 \cdot \text{kip} \cdot \text{in}$$





Section F.4.2:

Welded or unwelded:

$$S_{2_{F42}} := \begin{bmatrix} C_{br} \\ \hline 2.3 & \text{if Weld}_{F42} = 1 \\ \hline C_{brw} \\ \hline 2.3 & \text{otherwise} \end{bmatrix} = 29$$

$$s_{F42x} \coloneqq \frac{d}{t} \cdot \sqrt{\frac{L_{bx}}{d}} = 11.225 \qquad s_{F42y} \coloneqq \frac{t}{d} \cdot \sqrt{\frac{L_{by}}{t}} = 0.33$$

$$s_{F42y} := \frac{t}{d} \cdot \sqrt{\frac{L_{by}}{t}} = 0.33$$

Allowable flexural stress:

$$F_{b_F42x} := \frac{1}{\Omega_{by}} \cdot \left| \frac{\pi^2 \cdot E_a}{5.29 \cdot S_{F42x}^2} \right| \text{ if } S_{F42x} > S_{2_F42}$$

$$F_{b_F42x} = 30.083 \cdot ksi$$

otherwise
$$\begin{vmatrix} B_{br} - 2.3 \cdot D_{br} \cdot S_{F42x} & \text{if Weld}_{F42} = 1 \\ B_{brw} - 2.3 \cdot D_{brw} \cdot S_{F42x} & \text{otherwise} \end{vmatrix}$$

$$\begin{split} F_{b_F42y} &:= \frac{1}{\Omega_{by}} \cdot \left| \frac{\pi^2 \cdot E_a}{5.29 \cdot S_{F42y}^2} \right| & \text{if } S_{F42y} > S_{2_F42} \\ & \text{otherwise} \\ & \left| B_{br} - 2.3 \cdot D_{br} \cdot S_{F42y} \right| & \text{if } Weld_{F42} = 1 \\ & B_{brw} - 2.3 \cdot D_{brw} \cdot S_{F42y} & \text{otherwise} \end{split}$$

$$F_{b_{-}F42y} = 40.193 \cdot ksi$$

$$B_{br} - 2.3 \cdot D_{br} \cdot S_{F42y}$$
 if $Weld_{F42} = 1$
 $B_{brw} - 2.3 \cdot D_{brw} \cdot S_{F42y}$ otherwise

Allowable flexural capacity:

$$\mathbf{M}_{\mathbf{n}}_{\mathbf{F42x}} \coloneqq \mathbf{F}_{\mathbf{b}}_{\mathbf{F42x}} \cdot \mathbf{S}_{\mathbf{x}}$$

$$M_{n F42x} = 69097 \cdot lbf \cdot in$$

$$\mathsf{M}_{n_F42y} \coloneqq \mathsf{F}_{b_F42y} \cdot \mathsf{S}_{y}$$

$$M_{n_F42y} = 8792 \cdot lbf \cdot in$$

Flexural Summary:

$$M_{nx bar} := min(M_{n F812x}, M_{n F41x}, M_{n F42x})$$

$$M_{nx}$$
 bar = 63338·lbf·in

$$\mathbf{M_{ny_bar}} \coloneqq \min\!\!\left(\mathbf{M_{n_F812y}}, \!\mathbf{M_{n_F41y}}, \!\mathbf{M_{n_F42y}}\right)$$

$$M_{ny bar} = 6032 \cdot lbf \cdot in$$



Controlling flexural stress:

$$F_{bx_bar} := \frac{M_{nx_bar}}{S_x}$$

$$F_{bx bar} = 27.576 \cdot ksi$$

$$F_{by_bar} := \frac{M_{ny_bar}}{S_y}$$

 $F_{by_bar} = 27.576 \cdot ksi$

Shear capacity

Section G.2:

Welded or unwelded:

Slenderness limits:

$$S_{1_G2} := \begin{bmatrix} \frac{B_s - F_{sy}}{1.25 \cdot D_s} & \text{if Weld}_{G2} = 1 \\ \frac{B_{sw} - F_{syw}}{1.25 \cdot D_{sw}} & \text{otherwise} \end{bmatrix}$$

$$S_{2_G2} := \begin{bmatrix} \frac{C_s}{1.25} & \text{if Weld}_{G2} = 1 \\ \frac{C_{sw}}{1.25} & \text{otherwise} \end{bmatrix} = 63$$

d/t:

$$dt := \frac{d}{t} = 10.5$$

Allowable shear stress:

$$\begin{aligned} F_{v_G2} &\coloneqq \frac{1}{\Omega_v} \cdot \begin{vmatrix} \text{if Weld}_{G2} = 1 \\ F_{sy} & \text{if dt} < S_{1_G2} \\ \frac{1.3 \cdot \pi^2 \cdot E_a}{\left(1.25 \cdot \text{dt}\right)^2} & \text{if dt} > S_{2_G2} \\ B_s - 1.25 \cdot D_s \cdot \text{dt otherwise} \end{vmatrix} \end{aligned}$$

 $F_{v_G2} = 12.727 \cdot ksi$

otherwise



Allowable shear

capacity:
$$V_{n_G2} := F_{v_G2} \cdot d \cdot t$$

$$V_{n G2} = 33.409 \cdot kip$$

Shear Summary:

Controlling shear capacity:
$$V_{n \ bar} := V_{n \ G2}$$
 $V_{n \ bar} = 33.409 \cdot kip$

Controlling shear stress:
$$F_{v_bar} := \frac{V_{n_bar}}{A_{rea}}$$

$$F_{v_bar} = 12.727 \cdot ksi$$

Moment capacity:
$$M_{nx bar} = 63.338 \cdot \text{kip} \cdot \text{in}$$
 $F_{bx bar} = 27.576 \cdot \text{ksi}$

$$M_{ny_bar} = 6.032 \cdot kip \cdot in$$
 $F_{by_bar} = 27.576 \cdot ksi$

Shear capacity:
$$V_{n_bar} = 33.409 \cdot kip$$
 $F_{v_bar} = 12.727 \cdot ksi$

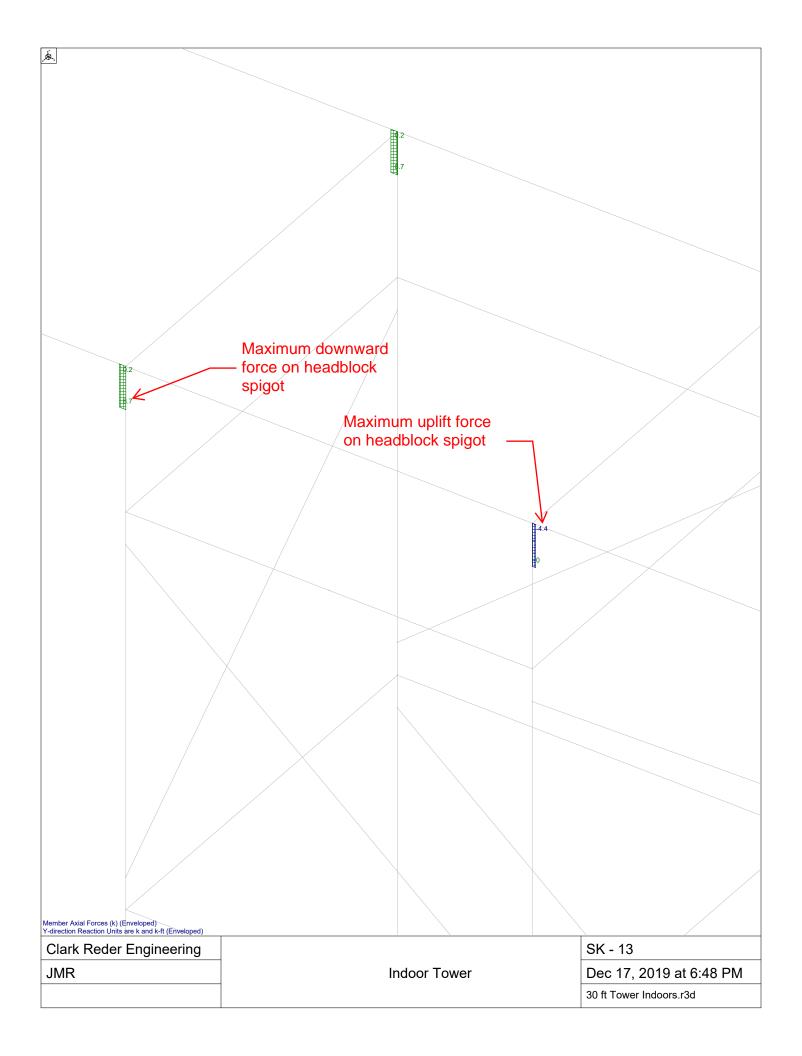
Allowable uplift load: uplift :=
$$\frac{M_{ny_bar}}{1.375 \cdot in} = 4387.052 \, lbf$$
 The plate can safely resist the uplift.

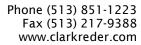
Consider bolts

Capacity of 5/8" grade 8 bolts:
$$V_{allow 58} := 10.4 \cdot kip$$

$$T_{\rm allow_58} := 17.3 \cdot kip$$

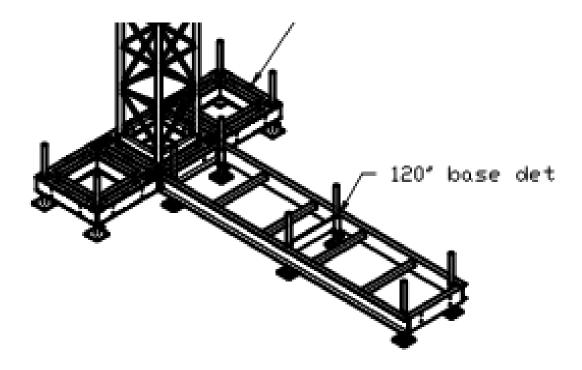
The maximum uplift is 4.4 kips. The maximum shear is minimal. The bolts are adequate.







Audio Tower Base





10 foot Base

Standard Beam AISC Parameters:

$$B := 1$$

$$span := 56 \cdot in$$

$$\mathbf{p}_{\mathbf{X}} := \begin{pmatrix} 5 \cdot \mathrm{kip} \\ 0 \cdot \mathrm{kip} \\ 0 \cdot \mathrm{kip} \\ 0 \cdot \mathrm{kip} \end{pmatrix} \mathbf{d}_{\mathbf{p}\mathbf{X}} := \begin{pmatrix} 0.5 \cdot \mathrm{span} \\ 20 \cdot \mathrm{ft} \\ 0 \cdot \mathrm{ft} \\ 0 \cdot \mathrm{ft} \end{pmatrix} \mathbf{w}_{1\mathbf{X}} := \begin{pmatrix} \mathbf{bm}_{\mathbf{W}t} \\ 0 \cdot \mathrm{klf} \\ 0 \cdot \mathrm{klf} \\ 0 \cdot \mathrm{klf} \end{pmatrix} \mathbf{d}_{\mathbf{W}1\mathbf{X}} := \begin{pmatrix} 0 \cdot \mathrm{ft} \\ 0 \cdot \mathrm{ft} \\ 0 \cdot \mathrm{ft} \\ 0 \cdot \mathrm{ft} \end{pmatrix} \mathbf{w}_{2\mathbf{X}} := \begin{pmatrix} \mathbf{bm}_{\mathbf{W}t} \\ 0 \cdot \mathrm{klf} \\ 0 \cdot \mathrm{klf} \\ 0 \cdot \mathrm{klf} \end{pmatrix} \mathbf{d}_{\mathbf{W}2\mathbf{X}} := \begin{pmatrix} \mathbf{bm}_{\mathbf{W}t} \\ 0 \cdot \mathrm{klf} \\ 0 \cdot \mathrm{klf} \\ 0 \cdot \mathrm{klf} \end{pmatrix}$$

$$\begin{pmatrix} \text{RL}_{x_B} \\ \text{RR}_{x_B} \\ \\ \text{Man}_{x_B} \end{pmatrix} := \begin{pmatrix} \text{R}_L \left(p_x, d_{px}, w_{1x}, w_{2x}, d_{w1x}, d_{w2x}, \text{span} \right) \\ \text{R}_R \left(p_x, d_{px}, w_{1x}, w_{2x}, d_{w1x}, d_{w2x}, \text{span} \right) \\ \text{Mom} \left(p_x, d_{px}, w_{1x}, w_{2x}, d_{w1x}, d_{w2x}, \text{span} \right) \end{pmatrix} \quad \text{RL}_{x_B} = 2.53 \cdot \text{kip} \quad \text{RR}_{x_B} = 2.53 \cdot \text{kip} \quad \text{M}_{x_B} = 5.869 \cdot \text{ft-kip}$$

Analysis Loads:

$$M_X := M_{X_{P}}$$

$$M_{\text{c}} := 0 \cdot \text{ft} \cdot \text{kip}$$

$$P_{\text{NAXA}} = 0 \cdot \text{kip}$$

$$M_X := M_{X_B}$$
 $M_X := 0 \cdot \text{ft} \cdot \text{kip}$ $M_X := 0 \cdot \text{kip}$ $M_X := Maximum \left(RR_{X_B}, RL_{X_B}\right)$

span

AISC Design Parameters:

$$F_{\text{W}} := 50 \cdot \text{ksi}$$
 $F_{\text{W}} := \text{span}$ $F_{\text{W}} := \text{span}$ $F_{\text{W}} := 2 \cdot \text{ft}$

$$L_{xx} = span$$

$$L_{loc} = 2 \cdot f$$

Member Specification: PROF := "C6X13"

$$A := \text{Results} \left(\text{PROF}, L_x, K_x, L_v, K_v, L_b, C_b, C_{mx}, C_{mv}, M_x, M_v, P_{ax}, V_x, \text{Ratio}, F_v, E \right)$$

$$A = \begin{pmatrix} "C6X13" & "PASS" & "fa_comp (ksi) = " & 0 & "Fa_comp (ksi) = " & 13.124 & "Ix (in^4) = " & 17.4 \\ "IntActRatio = " & 0.405 & "fbx (ksi) & = " & 12.142 & "Fbx (ksi) & = " & 30 & "Iy (in^4) = " & 1.05 \\ "Shear Ratio = " & 0.048 & "fby (ksi) & = " & 0 & "Fby (ksi) & = " & 30 & "Fy (ksi) = " & 50 \end{pmatrix}$$

$$\delta_{\text{location}} = \frac{\text{span}}{2}$$

$$\mathcal{M} := \text{Defl}\left(p_x, d_{px}, w_{1x}, w_{2x}, d_{w1x}, d_{w2x}, \text{span}, E, A_{1,8} \cdot \text{in}^4, \delta_{\text{location}}\right)$$

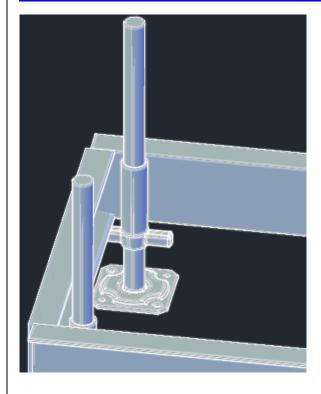
 $\delta = 0.037 \cdot in$

$$\frac{\text{span}}{s} = 1533.078$$





Vertical weld capacity of screw jack support



Determine the weld capacity of the vertical support:

Weld strength: $P_{weld} := 3 \cdot 0.92 \cdot \frac{kip}{in} \cdot 6 \cdot in \cdot 2 = 33.12 \cdot kip$

The screw jack has a miniumum capacity of 12,500#.

Moment and Shear Capacities of C6x13

Area of C6x13: $A_{c6} := 3.82 \cdot in^2$

Plastic section modulus of C6: $Zxc6 := 7.29 \cdot in^3$ $Zyc6 := 1.35 \cdot in^3$

Section modulus of C6: $Sxc6 := 5.78 \cdot in^3$ $Syc6 := 0.638 \cdot in^3$

Moment of inertia of C6: $Ixc6 := 17.3 \cdot in^4$ $Iyc6 := 1.05 \cdot in^4$

Radius of gyration: $rxc6 := 2.13 \cdot in$ $ryc6 := 0.524 \cdot in$

Material properties: F_w:= 50·ksi



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Yielding, F2.1: $M_n := F_y \cdot Zxc6 = 30.375 \cdot ft \cdot kip$

Lateral torsional buckling, F2.2:

Length between braced points: $L_{\text{Max}} = 2 \cdot \text{ft}$

 $\text{Limiting lateral unbraced length:} \qquad \text{L}_p \coloneqq 1.76 \cdot \text{ryc}6 \cdot \sqrt{\frac{E_s}{F_v}} = 22.21 \cdot \text{in}$

Lateral torsional buckling does not control.

Allowable moment in C6: $M_{a_c6} := \frac{M_n}{1.67} = 18.189 \cdot \text{ft} \cdot \text{kip}$

Weak Axis Bending

$$M_{ay_c6} := \frac{\min(F_y \cdot Zyc6, 1.6 \cdot F_y \cdot Syc6)}{1.67} = 2.547 \cdot \text{ft-kip}$$



Connection Tab

Safety factors used in analysis per AISC:

Bending: $\Omega_{\rm b} := 1.67$ Bearing: $\Omega_{\rm brg} := 2.0$ Pin shear: $\Omega_{\rm v} := 2.0$

Tension: $\Omega_{t} := 1.67$ Tensile rupture: $\Omega_{tr} := 2.0$ Shear yielding: $\Omega_{sy} := 1.5$

Compression: $\Omega_c := 1.67$ Shear rupture: $\Omega_{sr} := 2.0$ Block shear: $\Omega_{bs} := 2.0$

Plate height: $h_{pl} := 5 \cdot in$

Plate thickness: $t_{pl} := 0.5 \cdot in$

Plate length: $l_{pl} := 4 \cdot in$ Diameter of pin: $d_{pin} := 0.625 \cdot in$

Hole Diameter: $d_{hole} := 0.3438 \cdot in \cdot 2 = 0.688 \cdot in$ Pin yield strength: Fy_{pin} := 120 · ksi

Yield strength of plate: $Fy_{plate} := 36 \cdot ksi$ Pin tensile strength: $Fu_{pin} := 150 \cdot ksi$

Tensile strength of plate: Fuplate := 58·ksi

Pin shear capacity:

The pin will be in double shear. Use Table J3.2 for nominal stress of fasteners.

Nominal shear stress: $F_{nv pin} := 0.5 \cdot Fu_{pin}$ $F_{nv pin} = 75 \cdot ksi$

Area of pin: $A_{pin} := \frac{\pi \cdot d_{pin}^2}{4}$ $A_{pin} = 0.307 \cdot in^2$

Available double shear capacity of pin: $V_{a_pin} := \frac{F_{nv_pin} \cdot A_{pin}}{Q}$ $V_{a_pin} = 11.505 \cdot kip$

Bearing at Pin Hole

Clear distance between edge of hole and end of plate: $L_c := 1 \cdot in - \frac{d_{hole}}{2}$ $L_c = 0.656 \cdot in$

Available bearing strength:

 $Ra_{brg} := \frac{if \left(1.2 \cdot L_c \cdot t_{pl} \cdot Fu_{plate} \leq 2.4 \cdot d_{pin} \cdot t_{pl} \cdot Fu_{plate}, 1.2 \cdot L_c \cdot t_{pl} \cdot Fu_{plate}, 2.4 \cdot d_{pin} \cdot t_{pl} \cdot Fu_{plate}\right)}{\Omega_{brg}}$

 $Ra_{brg} = 11.418 \cdot kip$



Plate tension capacity

The plate will be checked for tensile yielding on gross section, tensile rupture on net section, shear yielding and shear rupture per section J4 of AISC.

$$\underline{\text{Tensile yielding:}} \qquad R_{ty} \coloneqq \frac{Fy_{plate} \cdot t_{pl} \cdot h_{pl}}{\Omega_t}$$

$$R_{ty} = 53.892 \cdot kip$$

$$\text{Effective area:} \qquad A_e := if \Bigg[t_{pl} \cdot \Bigg[h_{pl} - \Bigg(d_{hole} + \frac{1}{16} \cdot in \Bigg) \Bigg] \leq 0.85 \cdot \Big(t_{pl} \cdot h_{pl} \Big), \\ t_{pl} \cdot \Bigg[h_{pl} - \Bigg(d_{hole} + \frac{1}{16} \cdot in \Bigg) \Bigg], \\ 0.85 \cdot \Big(t_{pl} \cdot h_{pl} \Big) \Bigg] \leq 0.85 \cdot \Big(t_{pl} \cdot h_{pl} \Big), \\ t_{pl} \cdot \Bigg[h_{pl} - \Bigg(d_{hole} + \frac{1}{16} \cdot in \Bigg) \Bigg], \\ t_{pl} \cdot \Bigg[h_{pl} - \Bigg(d_{hole} + \frac{1}{16} \cdot in \Bigg) \Bigg], \\ t_{pl} \cdot \Bigg[h_{pl} - \Bigg(d_{hole} + \frac{1}{16} \cdot in \Bigg) \Bigg], \\ t_{pl} \cdot \Bigg[h_{pl} - \Bigg(d_{hole} + \frac{1}{16} \cdot in \Bigg) \Bigg], \\ t_{pl} \cdot \Bigg[h_{pl} - \Bigg(d_{hole} + \frac{1}{16} \cdot in \Bigg) \Bigg], \\ t_{pl} \cdot \Bigg[h_{pl} - \Bigg(d_{hole} + \frac{1}{16} \cdot in \Bigg) \Bigg], \\ t_{pl} \cdot \Bigg[h_{pl} - \Bigg(d_{hole} + \frac{1}{16} \cdot in \Bigg) \Bigg], \\ t_{pl} \cdot \Bigg[h_{pl} - \Bigg(d_{hole} + \frac{1}{16} \cdot in \Bigg) \Bigg], \\ t_{pl} \cdot \Bigg[h_{pl} - \Bigg(d_{hole} + \frac{1}{16} \cdot in \Bigg) \Bigg], \\ t_{pl} \cdot \Bigg[h_{pl} - \Bigg(d_{hole} + \frac{1}{16} \cdot in \Bigg) \Bigg], \\ t_{pl} \cdot \Bigg[h_{pl} - \Bigg(d_{hole} + \frac{1}{16} \cdot in \Bigg) \Bigg], \\ t_{pl} \cdot \Bigg[h_{pl} - \Bigg(d_{hole} + \frac{1}{16} \cdot in \Bigg) \Bigg], \\ t_{pl} \cdot \Bigg[h_{pl} - \Bigg(d_{hole} + \frac{1}{16} \cdot in \Bigg) \Bigg], \\ t_{pl} \cdot \Bigg[h_{pl} - \Bigg(d_{hole} + \frac{1}{16} \cdot in \Bigg) \Bigg], \\ t_{pl} \cdot \Bigg[h_{pl} - \Bigg(d_{hole} + \frac{1}{16} \cdot in \Bigg) \Bigg], \\ t_{pl} \cdot \Bigg[h_{pl} - \Bigg(d_{hole} + \frac{1}{16} \cdot in \Bigg) \Bigg], \\ t_{pl} \cdot \Bigg[h_{pl} - \Bigg(d_{hole} + \frac{1}{16} \cdot in \Bigg) \Bigg], \\ t_{pl} \cdot \Bigg[h_{pl} - \Bigg(d_{hole} + \frac{1}{16} \cdot in \Bigg) \Bigg], \\ t_{pl} \cdot \Bigg[h_{pl} - \Bigg(d_{hole} + \frac{1}{16} \cdot in \Bigg) \Bigg]$$

$$\underline{\text{Tensile rupture:}} \qquad R_{tr} := \frac{Fu_{plate} \cdot A_{e}}{\Omega_{tr}}$$

$$R_{tr} = 61.624 \cdot \text{kip}$$

Plate compression capacity

The nominal compressive strength shall be determined based on the limit state of flexural buckling.

Plate area:
$$A_{pl} := t_{pl} \cdot h_{pl}$$

Moment of inertia of plate:
$$Iy_{pl} := \frac{1}{12} \cdot h_{pl} \cdot t_{pl}^3$$

Radius of gyration of plate:
$$r_{y_pl} := \sqrt{\frac{Iy_{pl}}{A_{pl}}}$$

$$r_{y_pl} = 0.144 \cdot in$$

Effective buckling length of female plate:
$$L_{eff_pl} := 3 \cdot in$$

Effective length factor:
$$K_{pl} := 2$$

Slenderness:
$$kl_rpl := \frac{K_pl'L_{eff_pl}}{r_{v_pl}}$$
 $kl_rpl = 41.569$

Elastic critical buckling stress:
$$Fe_{pl} := \frac{\pi^2 \cdot E_s}{kl_{pl}^2}$$

$$Fe_{pl} = 165.636 \cdot ksi$$



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Flexural buckling stress:
$$\begin{aligned} & \text{Fer}_{plate} \coloneqq \begin{bmatrix} \frac{1 \text{ y}_{plate}}{\text{Fe}_{pl}} \\ 0.658 \end{bmatrix} \cdot \text{Fy}_{plate} \end{bmatrix} & \text{if } kl_r_{pl} \leq 4.71 \cdot \sqrt{\frac{E_s}{\text{Fy}_{plate}}} \\ & \left(0.877 \text{Fe}_{pl} \right) & \text{if } kl_r_{pl} > 4.71 \cdot \sqrt{\frac{E_s}{\text{Fy}_{plate}}} \end{aligned}$$

Available axial compressive strength:

$$P_{ca_plate} := \frac{Fcr_{plate} \cdot A_{pl}}{\Omega_{c}}$$

$$P_{ca\ plate} = 49.206 \cdot kip$$

Plate Flexural Capacity - AISC F11

Plastic section modulus:
$$Zx_{pl} := \frac{{}^{t}pl \cdot {}^{h}pl}{4}$$
 $Zx_{pl} = 3.125 \cdot in^{3}$

$$Zy_{pl} := \frac{t_{pl}^2 \cdot h_{pl}}{4}$$
 $Zy_{pl} = 0.313 \cdot in^3$

Elastic section modulus:
$$Sx_{pl} := \frac{t_{pl} \cdot h_{pl}^2}{6}$$
 $Sx_{pl} = 2.083 \cdot in^3$

$$Sy_{pl} := \frac{t_{pl}^2 \cdot h_{pl}}{6}$$
 $Sy_{pl} = 0.208 \cdot in^3$

Unbraced length of plate: Line 3 · in

Vertical flexural capacity:

$$\begin{aligned} & \text{Eqn F11-1:} \quad \text{M}_{px} \coloneqq \text{if} \Big(\text{Fy}_{plate} \cdot \text{Zx}_{pl} \leq 1.6 \cdot \text{Sx}_{pl} \cdot \text{Fy}_{plate}, \text{Fy}_{plate} \cdot \text{Zx}_{pl}, 1.6 \cdot \text{Sx}_{pl} \cdot \text{Fy}_{plate} \Big) \\ & \qquad \qquad \text{M}_{px} = 112.5 \cdot \text{in} \cdot \text{kip} \\ & \qquad \qquad \text{Chov} = 1.0 \end{aligned}$$

$$\textbf{Eqn F11-2:} \quad \mathbf{M_{n_ltb}} \coloneqq \mathbf{C_b} \cdot \left[1.52 - 0.274 \cdot \left(\frac{\mathbf{L_b \cdot h_{pl}}}{t_{pl}^2} \right) \cdot \frac{\mathbf{Fy_{plate}}}{\mathbf{E_s}} \right] \cdot \mathbf{Sx_{pl} \cdot Fy_{plate}}$$

$$\mathbf{M}_{\mathbf{nx_ltb}} \coloneqq \mathrm{if} \left(\mathbf{M}_{\mathbf{n_ltb}} \le \mathbf{M}_{\mathbf{px}}, \mathbf{M}_{\mathbf{n_ltb}}, \mathbf{M}_{\mathbf{px}} \right)$$

$$\mathbf{M}_{\mathbf{nx_ltb}} = 112.469 \cdot \mathrm{in} \cdot \mathrm{kip}$$



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$$F_{cr} := \frac{1.9 \cdot E_s \cdot C_b}{\left(\frac{L_b \cdot h_{pl}}{t_{pl}}\right)}$$

$$M_{nx s} = 112.5 \cdot in \cdot kip$$

Plate available flexural strength:
$$\begin{aligned} M_{nx_pl} &:= & M_{px} \quad \mathrm{if} \quad \frac{L_b \cdot h_{pl}}{t_{pl}^2} \leq \frac{0.08 \cdot E_s}{Fy_{plate}} \\ M_{nx_ltb} \quad \mathrm{if} \quad \frac{L_b \cdot h_{pl}}{t_{pl}^2} > \frac{0.08 \cdot E_s}{Fy_{plate}} \wedge \frac{L_b \cdot h_{pl}}{t_{pl}^2} \leq \frac{1.9 \cdot E_s}{Fy_{plate}} \\ M_{nx_s} \quad \mathrm{if} \quad \frac{L_b \cdot h_{pl}}{t_{pl}^2} \leq \frac{1.9 \cdot E_s}{Fy_{plate}} \end{aligned}$$

$$M_{nx pl} = 112.5 \cdot in \cdot kip$$

Available flexural strength:
$$M_{ax_pl} := \frac{M_{nx_pl}}{\Omega_b}$$

$$M_{ax_pl} = 5.614 \cdot \text{ft} \cdot \text{kip}$$

Allowable vertical shear:
$$P_{vert_pl} := \frac{M_{ax_pl}}{3 \cdot in}$$

$$P_{\text{vert pl}} = 22.455 \cdot \text{kip}$$

Horizontal flexural capacity

For rectangular bars bent about their rectangular axis, the limit state of lateral torsional buckling does not apply.

Available flexural strength:
$$M_{ay_pl} := \frac{M_{py}}{\Omega_{L}}$$
 $M_{ay_pl} = 6.737 \cdot in \cdot kip$





30"x30" Base

Moment and Shear Capacities of Base Channel (built up member)

Area of built up C: $A_{bub} := 4.5 \cdot in^2$

Plastic section modulus of channel: $Zxbub := 8.625 \cdot in^3$ $Zybub := 1.78 \cdot in^3$

Section modulus of channel: $Sxbub := 6.792 \cdot in^3$ $Sybub := 0.9485 \cdot in^3$

Moment of inertia of channel: Ixbub := $20.38 \cdot \text{in}^4$ Iybub := $1.344 \cdot \text{in}^4$

Radius of gyration: $rxbub := 2.128 \cdot in$ $rybub := 0.5465 \cdot in$

Depth of channel: $d_{bub} := 6 \cdot in$ $t_{fi} := 0.5 \cdot in$ $t_{fi} := 0.5 \cdot in$ $t_{fi} := 0.5 \cdot in$

Material properties: $F_{\text{WW}} = 36 \cdot \text{ksi}$

Yielding, F2.1: $M_{\text{Ma}} := F_{\text{V}} \cdot Zxbub = 25.875 \cdot \text{ft} \cdot \text{kip}$

Lateral torsional buckling, F2.2:

Length between braced points: $L_{h} := 1.33 \cdot ft$

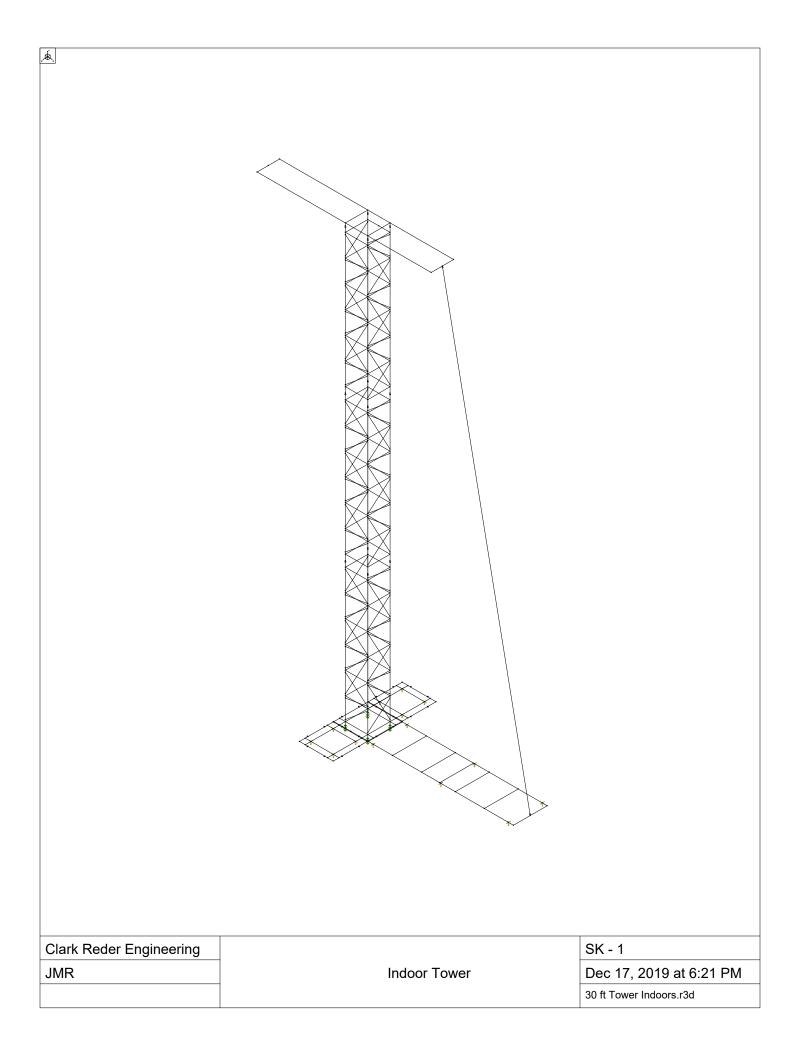
Lateral torsional buckling does not control.

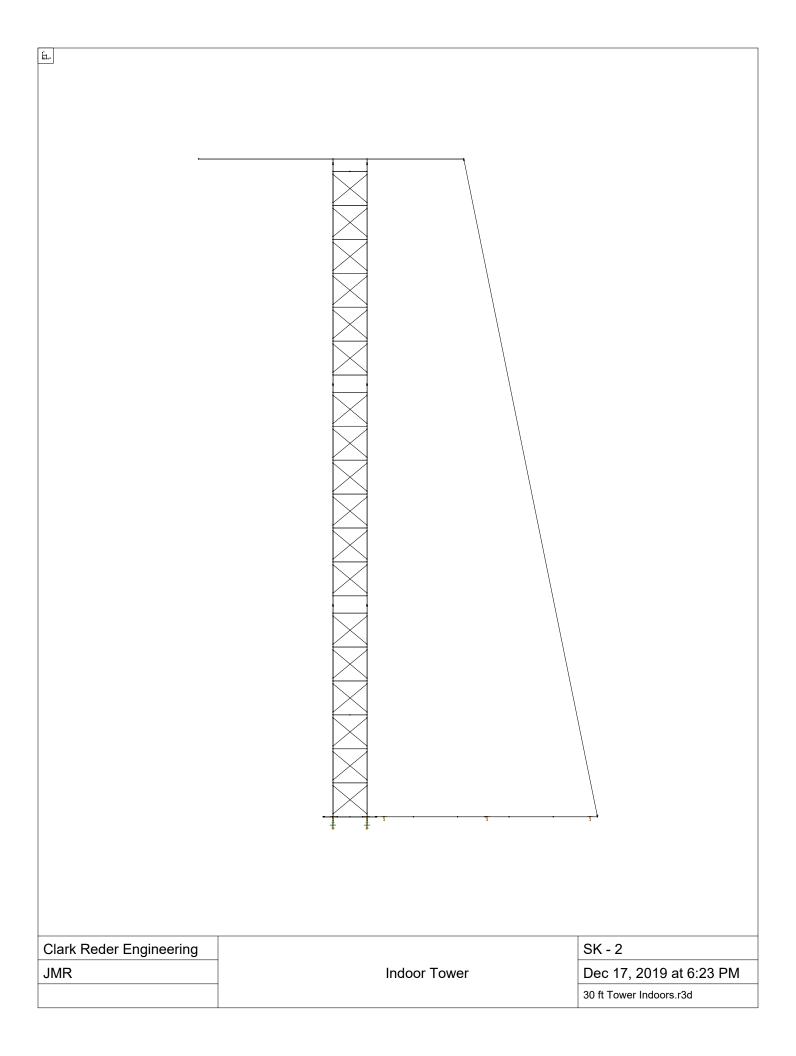
Allowable moment in channel: $M_{a_bub} := \frac{M_n}{1.67} = 15.494 \cdot \text{ft} \cdot \text{kip}$

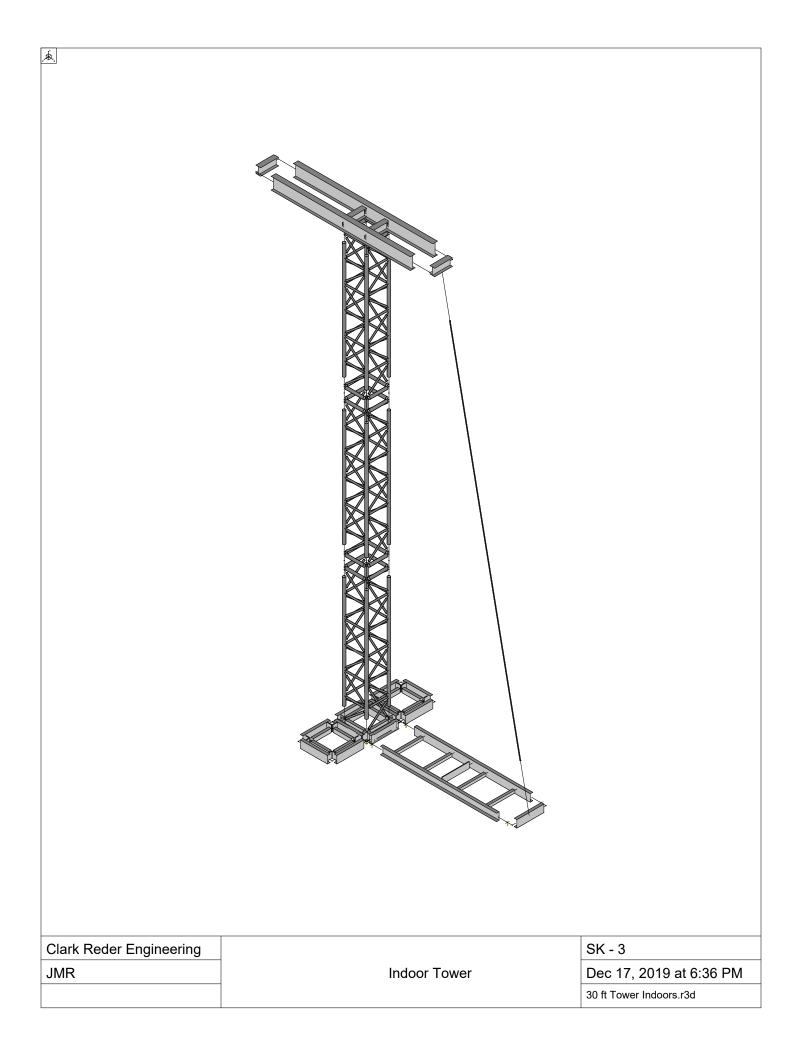


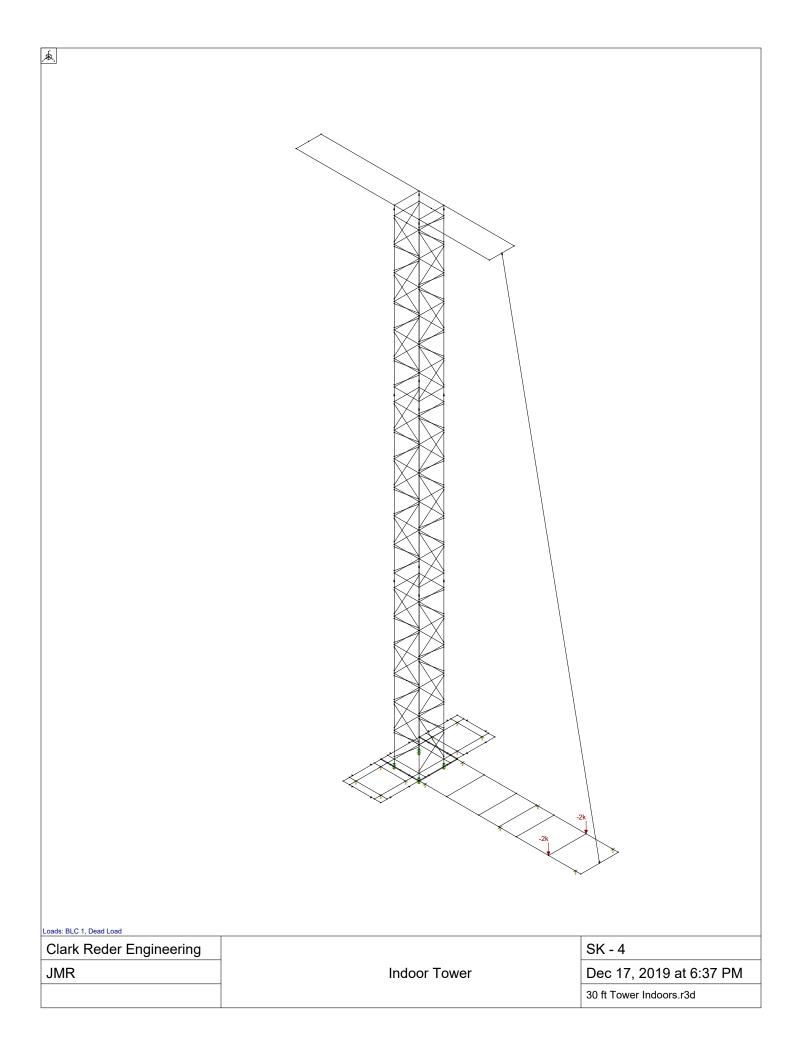
Weak Axis Bending

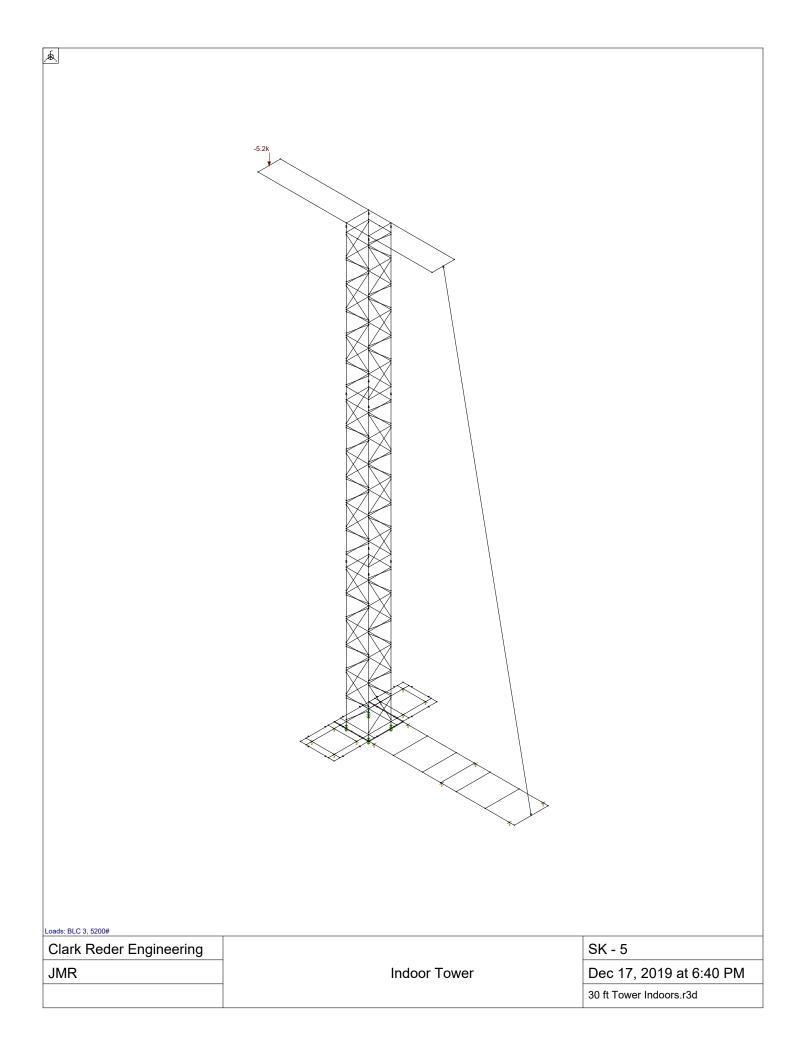
$$M_{ay_bub} \coloneqq \frac{min\!\!\left(F_y \cdot Zybub, 1.6 \cdot F_y \cdot Sybub\right)}{1.67} = 2.726 \cdot ft \cdot kip$$

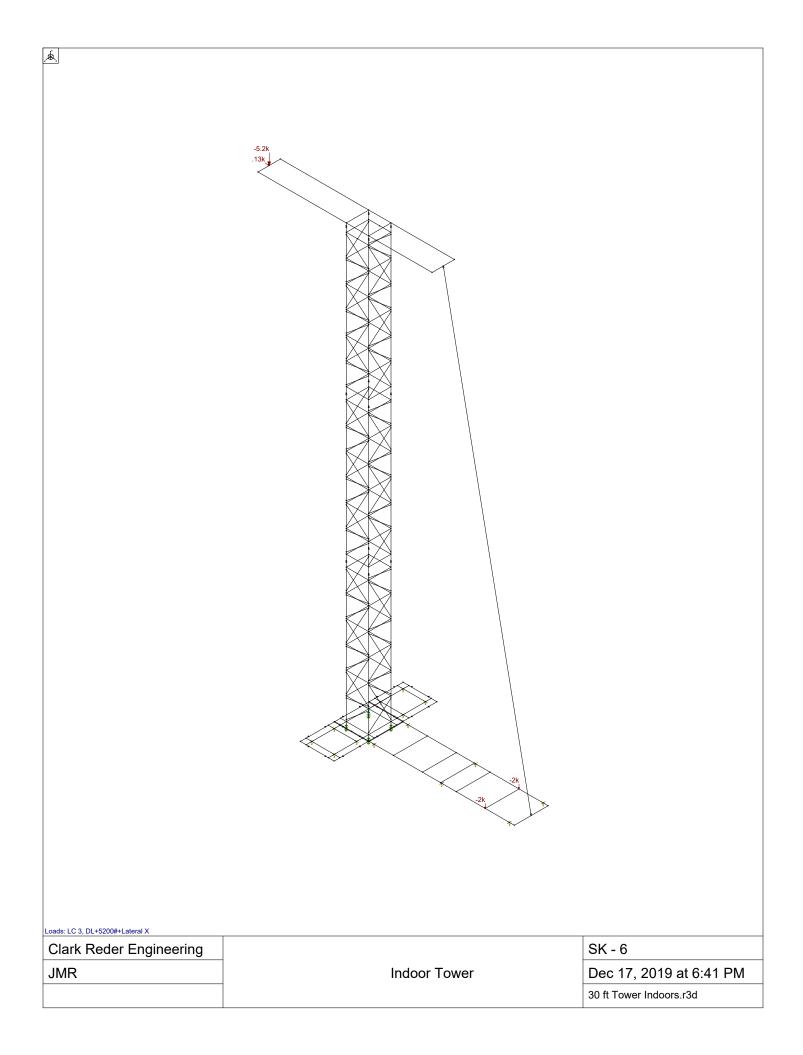


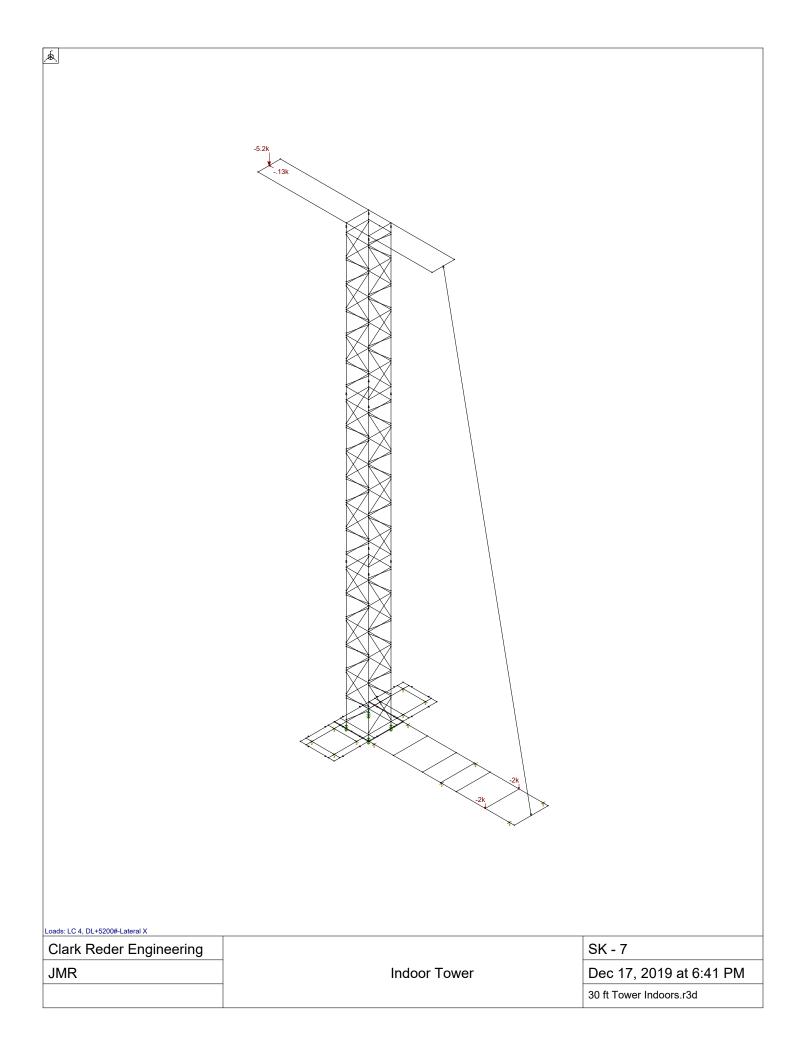


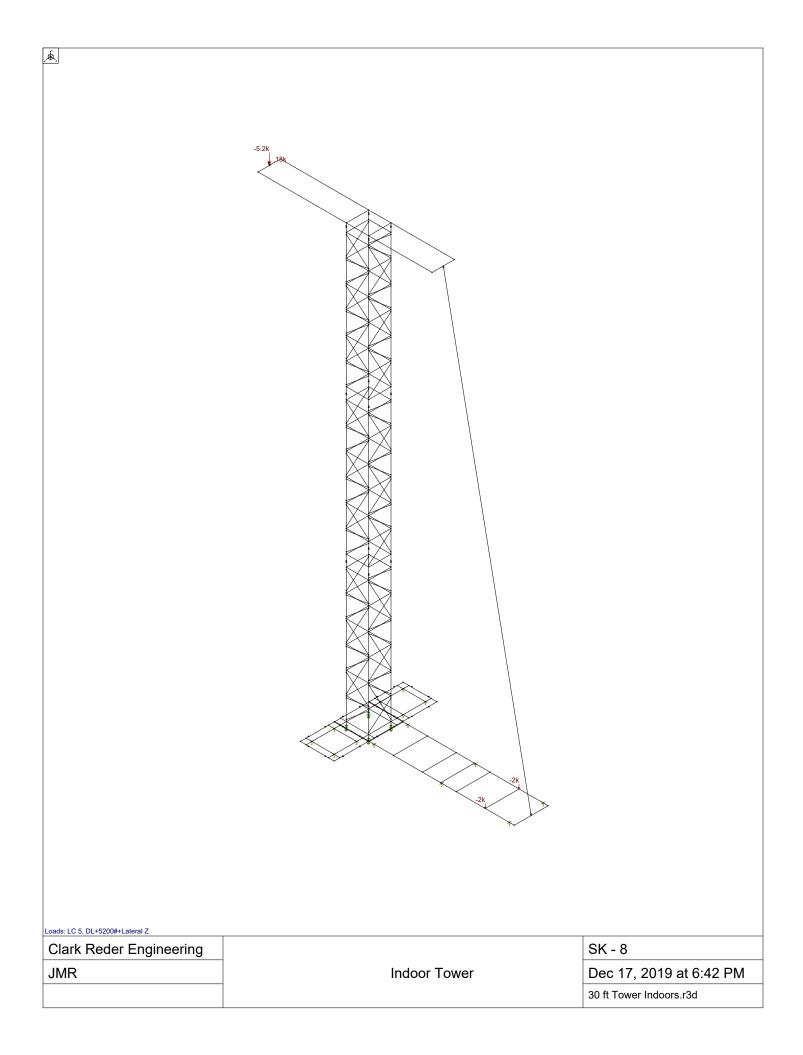


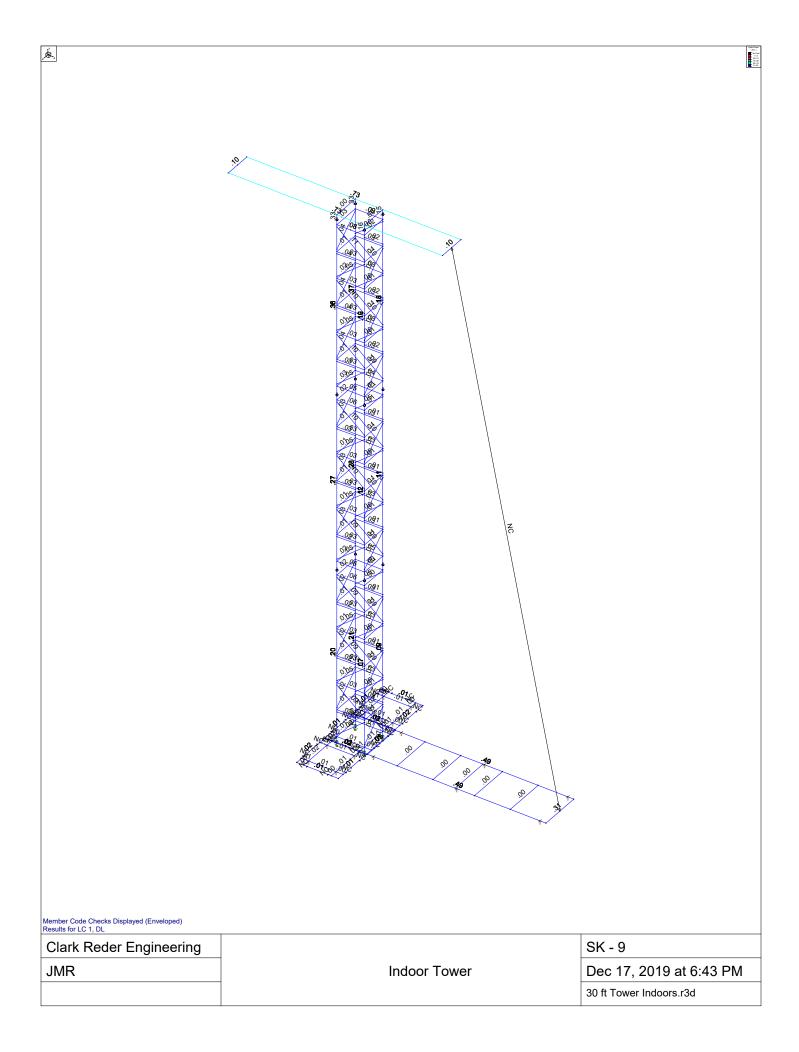


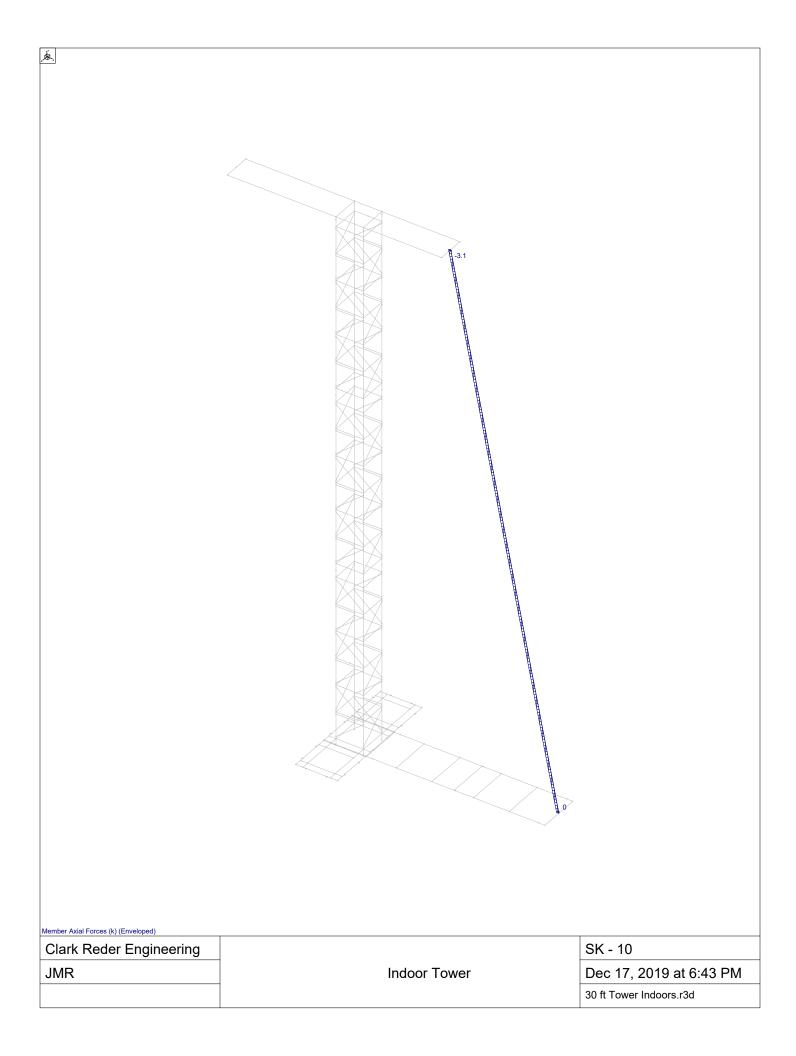


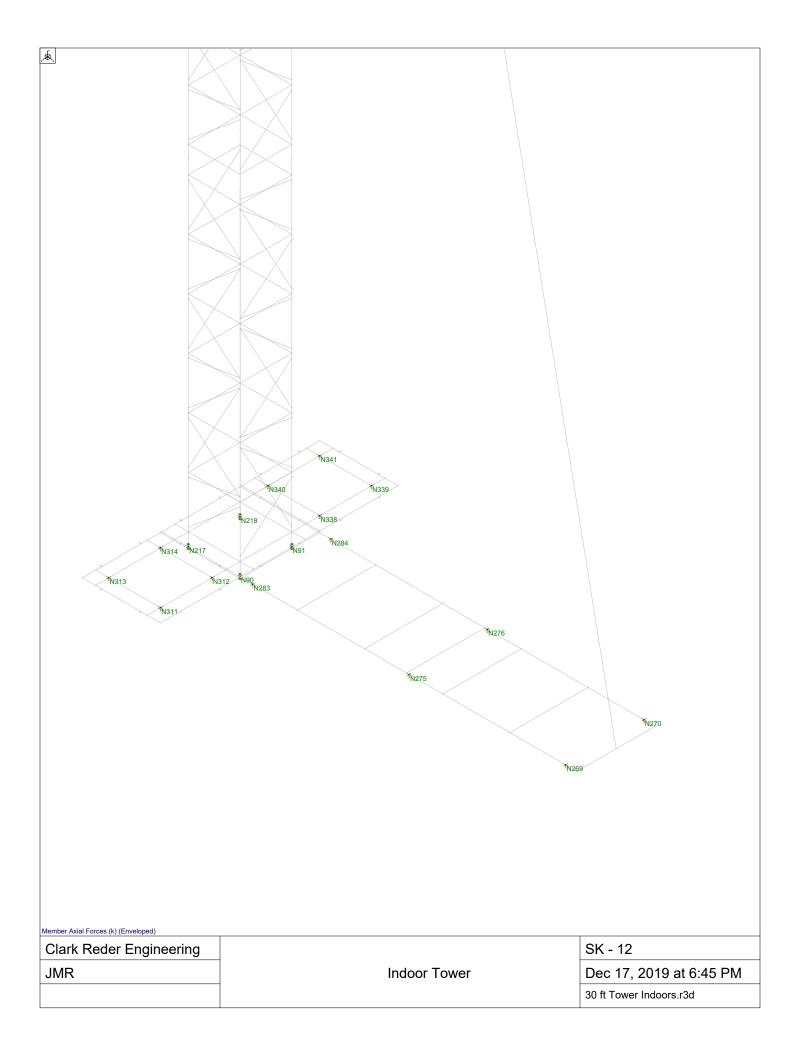












: Clark Reder Engineering: JMR

Indoor Tower

Dec 17, 2019 6:47 PM Checked By:__

Basic Load Cases

	BLC Description	Category	X Gravity	Y Gravity	Z Gravity	Joint	Point	Distribut	Area(M	Surface(
1	Dead Load	DĽ		-1		2			,	,
3	5200#	None				1				
6	Lateral 5200# X	None				1				
7	Lateral 5200# Z	None				1				

Load Combinations

	Description	So	P	S	BLC	Fac.	BLC	Fac	BLC	Fac	BLC	Fac	BLC	Fac.	BLC	Fac								
1	DĹ	Yes			1	1																		
2	DL+5200#	Yes			1	1	3	1																
3	DL+5200#+Lateral X				1	1	3	1	6	.025														
4	DL+5200#-Lateral X				1	1	3	1	6	025														
5	DL+5200#+Lateral Z				1	1	3	1	7	.025														
6	DL+5200#-Lateral Z				1	1	3	1	7	025														

Joint Reactions

	LC	Joint Label	X [k]	<u>Y [k]</u>	Z [k]	MX [k-ft]	MY [k-ft]	MZ [k-ft]
1	1	N90	009	.166	002	0	0	0
2	1	N217	002	.32	002	0	0	0
3	1	N91	01	.163	.002	0	0	0
4	1	N218	003	.32	.002	0	0	0
5	1	N269	.001	1.184	002	0	0	0
6	1	N270	.001	1.184	.002	0	0	0
7	1	N275	0	1.094	0	0	0	0
8	1	N276	0	1.094	0	0	0	0
9	1	N283	.004	0	.003	0	0	0
10	1	N284	.003	0	003	0	0	0
11	1	N311	0	.047	0	0	0	0
12	1	N312	.001	.04	0	0	0	0
13	1	N313	0	.039	002	0	0	0
14	1	N314	.006	.098	0	0	0	0
15	1	N338	.004	.047	0	0	0	0
16	1	N339	0	.042	002	0	0	0
17	1	N340	.005	.096	0	0	0	0
18	1	N341	0	.04	.002	0	0	0
19	1	Totals:	0	5.977	0			
20	1	COG (in):	X: 291.334	Y: 45.735	Z: 18.01			
21	2	N9Ò ´	105	1.211	017	0	0	0
22	2	N217	069	2.727	03	0	0	0
23	2	N91	101	1.206	.014	0	0	0
24	2	N218	069	2.735	.032	0	0	0
25	2	N269	.307	0	457	0	0	0
26	2	N270	.307	0	.457	0	0	0
27	2	N275	0	.331	0	0	0	0
28	2	N276	0	.331	0	0	0	0
29	2	N283	04	.715	033	0	0	0
30	2	N284	048	.723	.045	0	0	0
31	2	N311	0	0	007	0	0	0
32	2	N312	02	.149	008	0	0	0
33	2	N313	.01	0	.027	0	0	0
34	2	N314	078	.455	015	0	0	0
35	2	N338	051	.144	013	0	0	0
36	2	N339	.002	0	.02	0	0	0
37	2	N340	056	.45	.004	0	0	0



Company Designer Job Number Model Name

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Joint Reactions (Continued)

	LC	Joint Label	X [k]	Y [k]	Z [k]	MX [k-ft]	MY [k-ft]	MZ [k-ft]
38	2	N341	.011	0	018	0	0	0
39	2	Totals:	0	11.177	0			
40	2	COG (in):	X: 216.623	Y: 192.883	Z: 18.006			